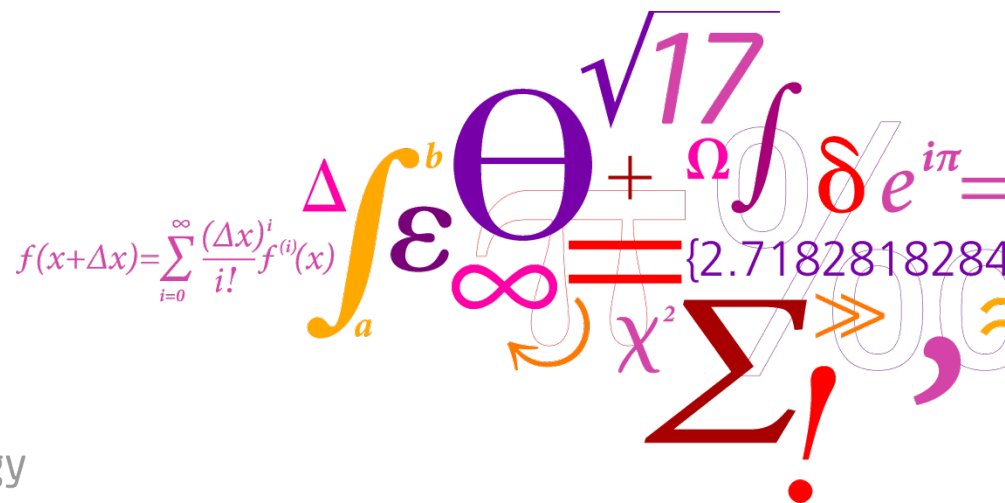


Comparison of the Wake of Different Kinds of Wind Turbine CFD Models

Niels Trolborg, Frederik Zahle, Pierre-Elouan Réthoré, Niels N. Sørensen

Wind Energy Department, DTU Wind Energy, DK-4000 Roskilde, Denmark



Background

Wind turbine models in CFD

- Fully resolved rotor (FR)
- Actuator line model (AL)
- Actuator disc model (AD)

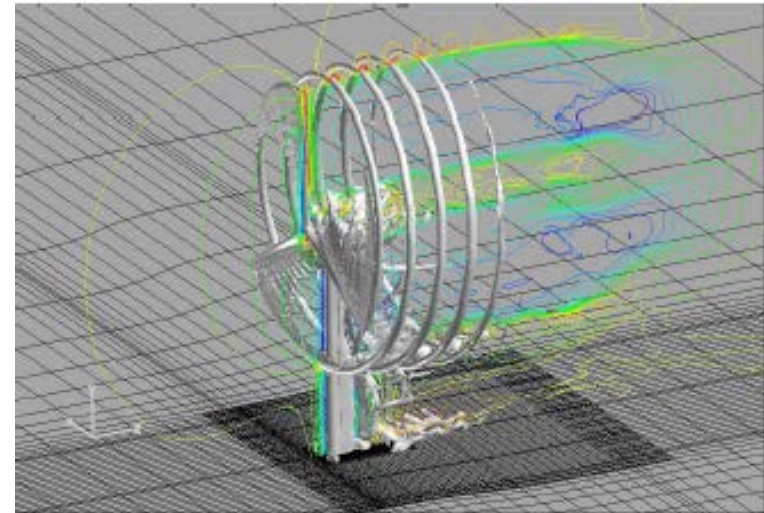
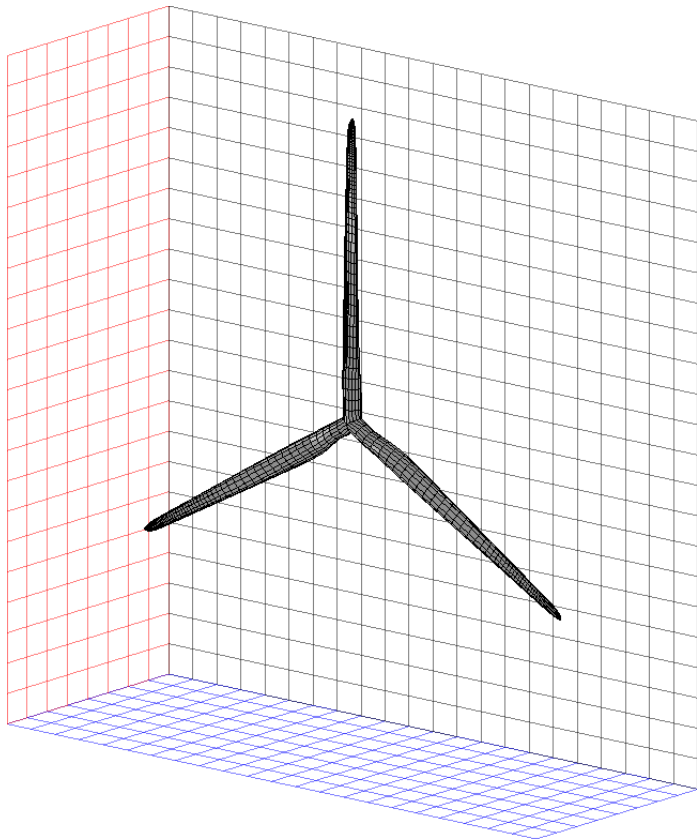


Background

Wind turbine models in CFD

- Fully resolved rotor (FR)
- Actuator line model (AL)
- Actuator disc model (AD)

- The blade/airfoil boundary layer is resolved
- The required number of grid points for one rotor using RANS is $O(10^7)$
- Provides detailed insight about flow behaviour
- Usually used for accurately predict loads and power production
- Too computationally heavy for several wind turbines.

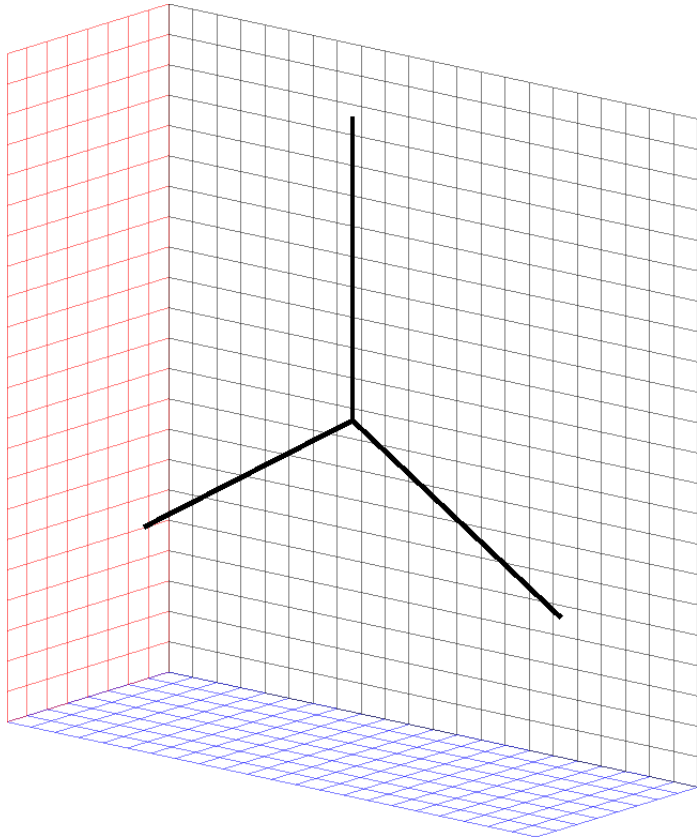


Background

Wind turbine models in CFD

- Fully resolved rotor (FR)
- Actuator line model (AL)
- Actuator disc model (AD)

➤ Blades represented as lines.

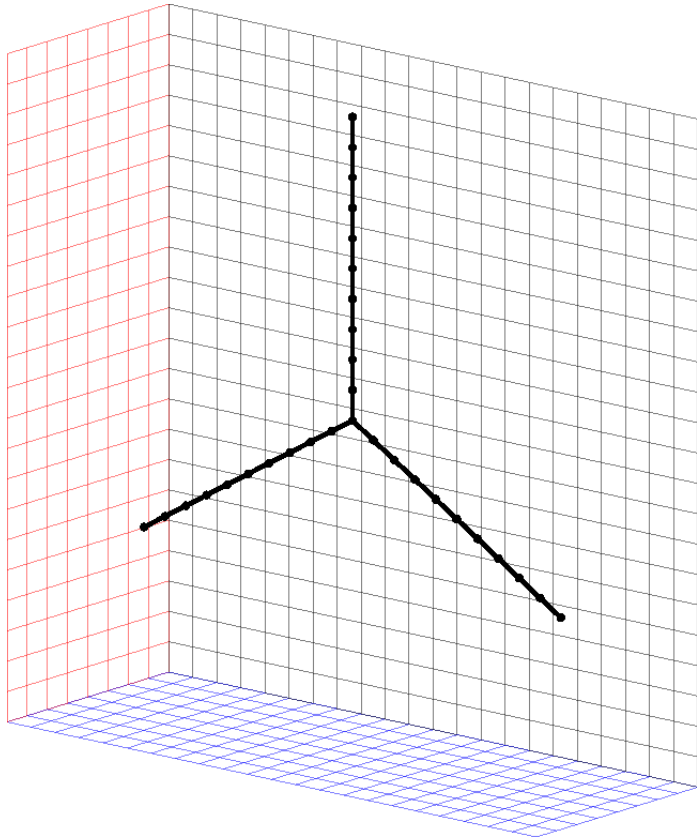


Background

Wind turbine models in CFD

- Fully resolved rotor (FR)
- Actuator line model (AL)
- Actuator disc model (AD)

- Blades represented as lines.
- Aerodynamic blade forces determined from 2D airfoil data.

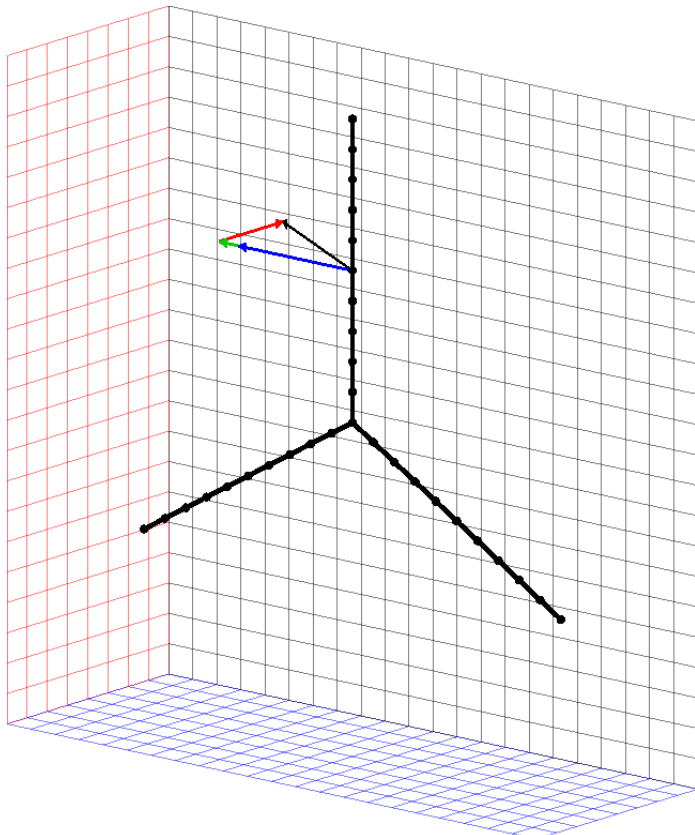


Background

Wind turbine models in CFD

- Fully resolved rotor (FR)
- Actuator line model (AL)
- Actuator disc model (AD)

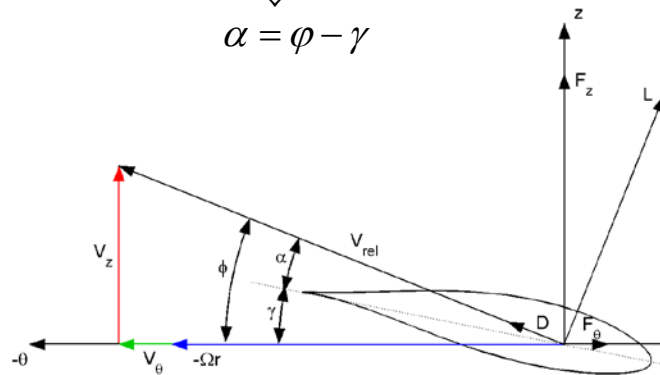
- Blades represented as lines.
- Aerodynamic blade forces determined from 2D airfoil data.



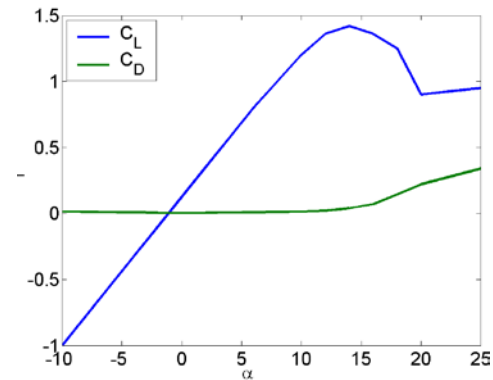
$$\varphi = \tan^{-1}\left(\frac{V_z}{\Omega r - V_\theta}\right)$$

$$\Downarrow$$

$$\alpha = \varphi - \gamma$$



$$\mathbf{f} = \begin{pmatrix} L \\ D \end{pmatrix} = \frac{1}{2} \rho V_{rel}^2 c \begin{pmatrix} C_L(\alpha) \mathbf{e}_L \\ C_D(\alpha) \mathbf{e}_D \end{pmatrix}$$



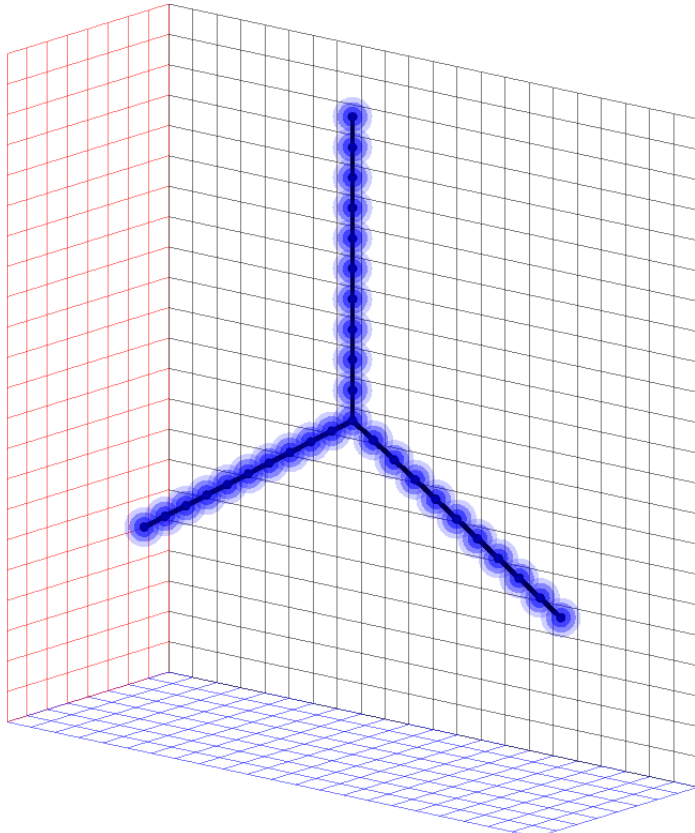
Background

Wind turbine models in CFD

- Fully resolved rotor (FR)
- Actuator line model (AL)
- Actuator disc model (AD)

- Blades represented as lines.
- Aerodynamic blade forces determined from 2D airfoil data.
- Blade forces smeared to avoid singular behaviour.

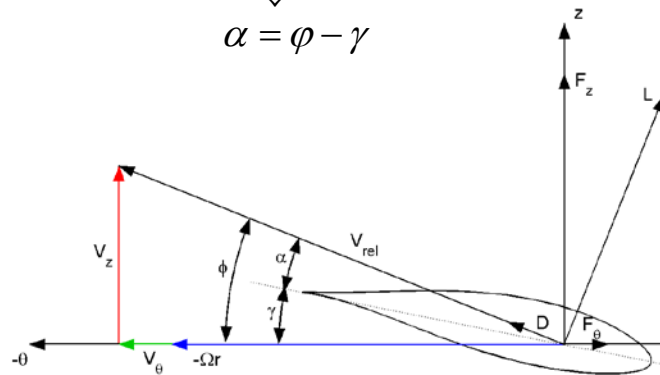
$$\mathbf{f}_\varepsilon = \mathbf{f} \otimes \eta_\varepsilon, \quad \eta_\varepsilon = \frac{1}{\varepsilon^3 \pi^{3/2}} \exp\left[-\frac{d^2}{\varepsilon^2}\right]$$



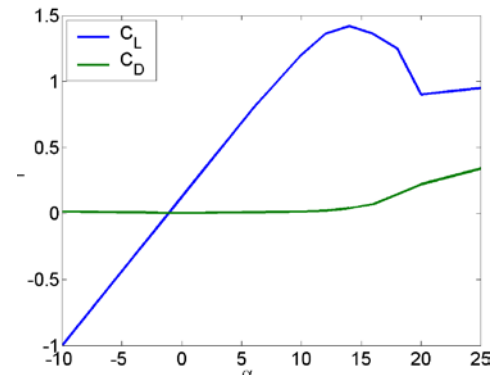
$$\varphi = \tan^{-1}\left(\frac{V_z}{\Omega r - V_\theta}\right)$$

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$$\alpha = \varphi - \gamma$$



$$\mathbf{f} = \begin{pmatrix} L \\ D \end{pmatrix} = \frac{1}{2} \rho V_{rel}^2 c \begin{pmatrix} C_L(\alpha) \mathbf{e}_L \\ C_D(\alpha) \mathbf{e}_D \end{pmatrix}$$



Wind turbine models in CFD

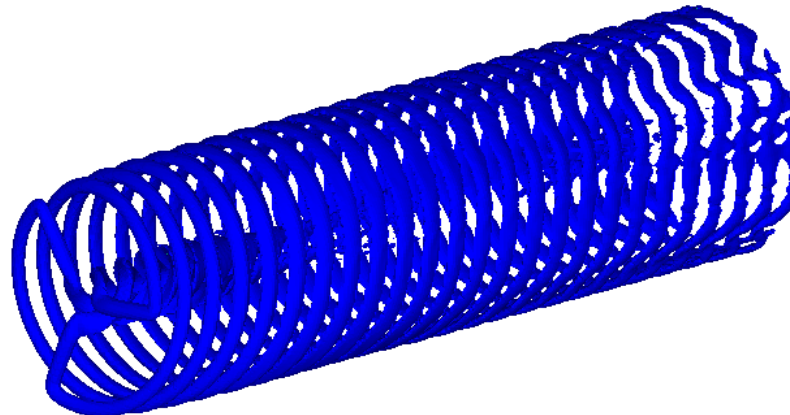
- Fully resolved rotor (FR)
- Actuator line model (AL)
- Actuator disc model (AD)

Advantages:

- Low number of grid points $O(10^6)$ needed compared to full rotor CFD.
- Applicable with simple grid geometries.
- Captures the most important features of the wake including tip/root vortices.
- Well suited for LES simulations (no boundary layers need to be resolved)

Disadvantages:

- Relies on airfoil data

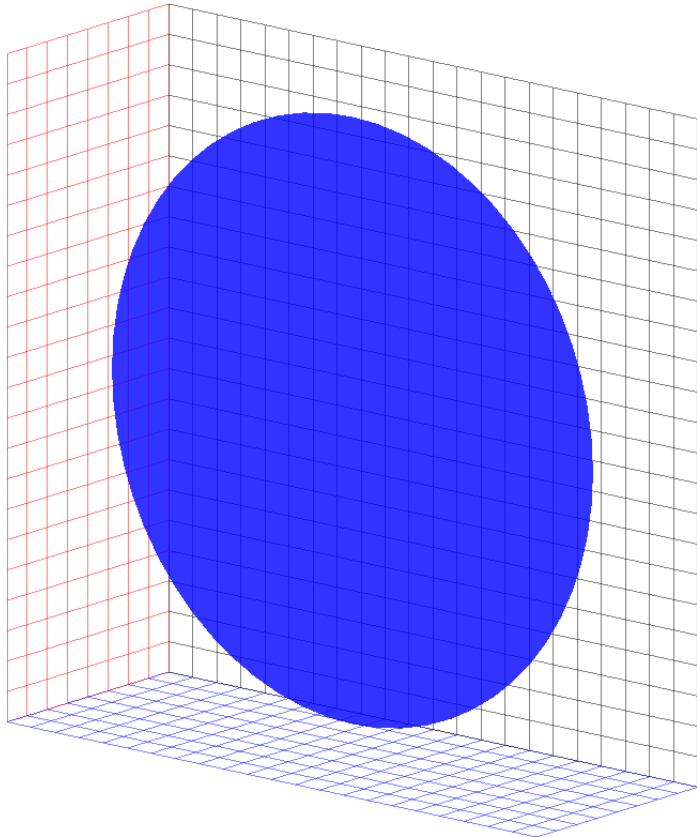


Background

Wind turbine models in CFD

- Fully resolved rotor (FR)
- Actuator line model (AL)
- Actuator disc model (AD)

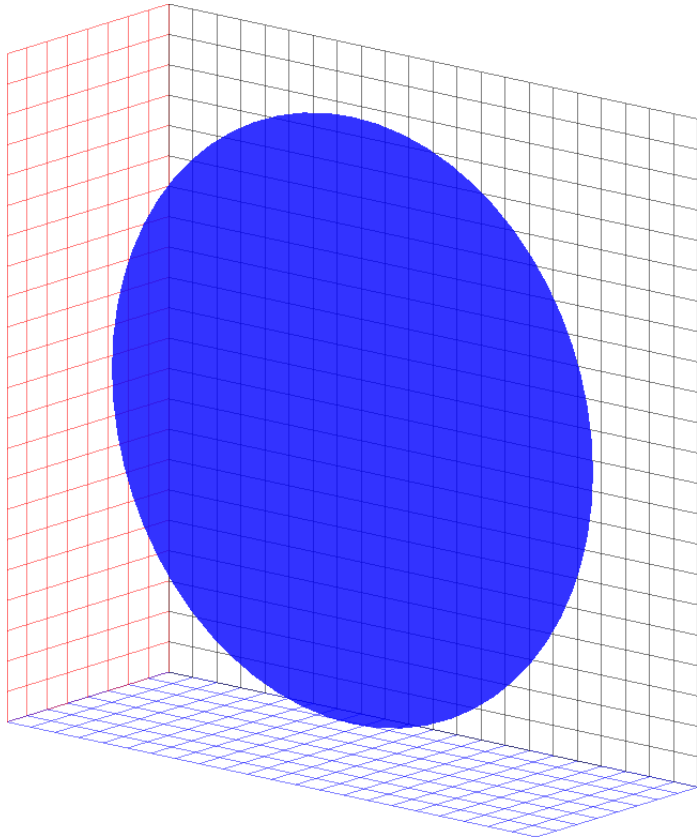
➤ Rotor represented by forces distributed on permeable disc.



Wind turbine models in CFD

- Fully resolved rotor (FR)
- Actuator line model (AL)
- Actuator disc model (AD)

- Rotor represented by forces distributed on permeable disc.
- The disc loading is either prescribed or determined from airfoil data.
- Modified Rhie-Chow algorithm to avoid pressure velocity decoupling with body forces



Wind turbine models in CFD

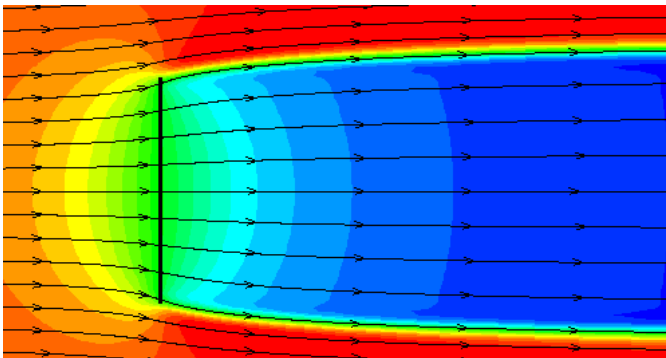
- Fully resolved rotor (FR)
- Actuator line model (AL)
- Actuator disc model (AD)

Advantages:

- Low number of grid points
- Applicable with simple grid geometries
- Well suited for LES simulations
- Large time steps can be used
- Can run in steady state

Disadvantages:

- Relies on airfoil data
- Does not capture influence of individual blades
- May be questionable in non-uniform inflow



Axial velocity contours and streamlines for a uniformly loaded disc at $C_T=0.89$

Background & Objectives

Summary:

- AL/AD typically used for wake studies
- Details of rotor geometry assumed unimportant in far wake

Objectives:

- Study importance of wind turbine model on wake characteristics
 - How much details are lost due to the simpler models?
- Conduct a consistent comparison of the three models
 - Simulate wake of the NREL 5MW Reference turbine
 - Same numerical setup for all models



Approach – Flow solver

EllipSys3D:

- In-house CFD code
- Incompressible Navier-Stokes equations
- Finite volume discretization
- Structured curvilinear grids
- Pressure/Velocity formulation
- Multigrid
- Multiblock
- Grid sequencing
- MPI

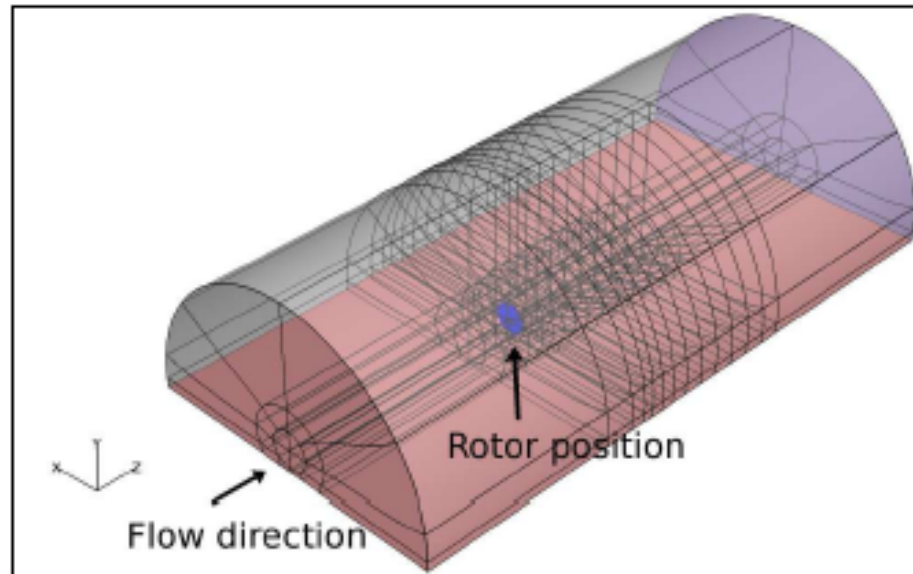
Solver parameters:

- QUICK/QUICK_CDS4
- SIMPLE

Approach - Numerical setup

Background mesh:

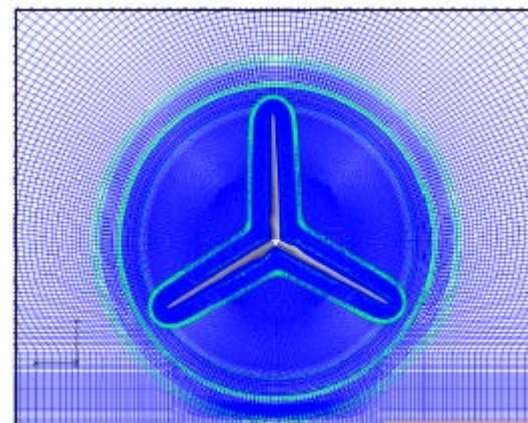
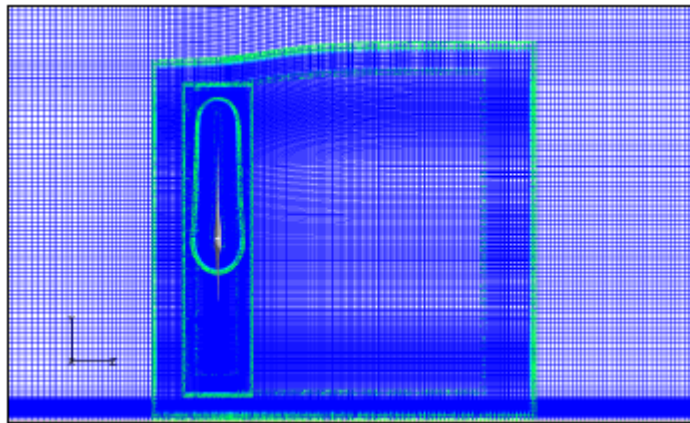
- Same background mesh for all simulations
- Half cylinder with radius $10R$
- 308 blocks of 32^3 ($10.1 \cdot 10^6$ cells)
- High resolution of the first $5D$ of the wake (80 cells per rotor diameter)



Approach - Numerical setup

Full rotor with overset grid:

- Four overlapping mesh groups
- Rotor mesh generated using HypGrid3D to form an O-O topology
- Total number of grid points is $23 \cdot 10^6$
- Rotor surface with a non-slip boundary condition
- First cell height $y = 1.0 \cdot 10^{-6}$ ($y^+ < 2$)



Approach - Numerical setup

Actuator line simulations:

- Same background mesh as the full rotor simulation ($10.1 \cdot 10^6$ cells)
- Force smearing using 3D convolution
- 33 force elements along each line



Approach - Numerical setup

Actuator disc simulations:

- Same background mesh as the full rotor simulation ($10.1 \cdot 10^6$ cells)
- 33 radial force elements
- Modified Rhie-Chow algorithm to avoid pressure velocity decoupling with body forces



Results

Test cases

- Uniform inflow (RANS)
- Uniform inflow (DES)
- Turbulent inflow (DES)

Test cases

- Uniform inflow (RANS)
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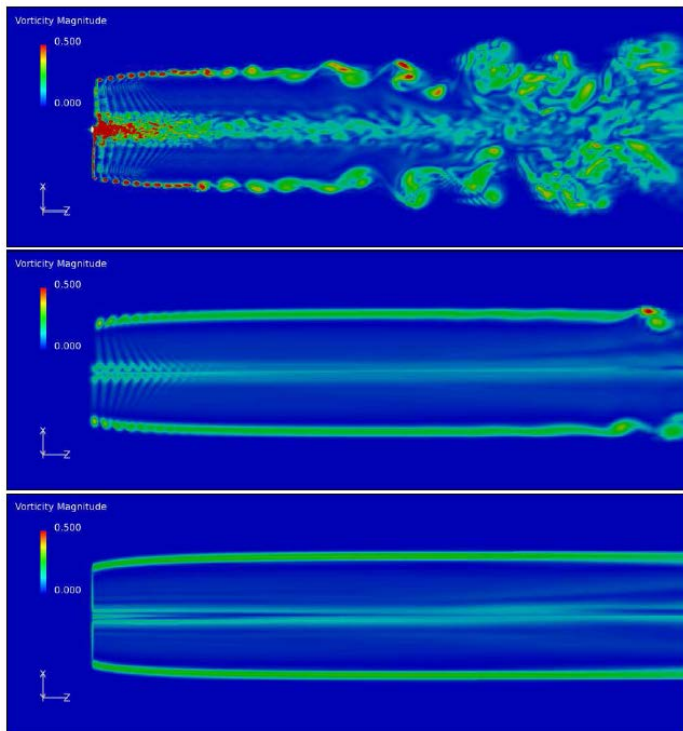
- $V_\infty = 8 \text{ m/s}$
- Time step $\Delta t = 0.00435\text{s}$, 0.0244s and 0.0600s in FR, AL and AD, respectively (1500, 267 and 109 time steps per rev.)
- Forces from FR simulation applied directly in the AL and AD simulations
- Computing expenses:
 - $T_{\text{FR}} \approx 21T_{\text{AL}} \approx 45T_{\text{AD}}$

Results

Test cases

- Uniform inflow (RANS)
- Uniform inflow (DES)
- Turbulent inflow (DES)

- Vorticity from tip vortices much stronger in FR than in AL and AD.
- Wake of FR becomes unstable and breaks up.
- Similar vorticity contours for AL and AD (except for instability in the far wake of AL)
- Reasons for more unstable wake of FR:
 - Higher grid resolution
 - Fluctuating loads (e.g. stall effects near root)



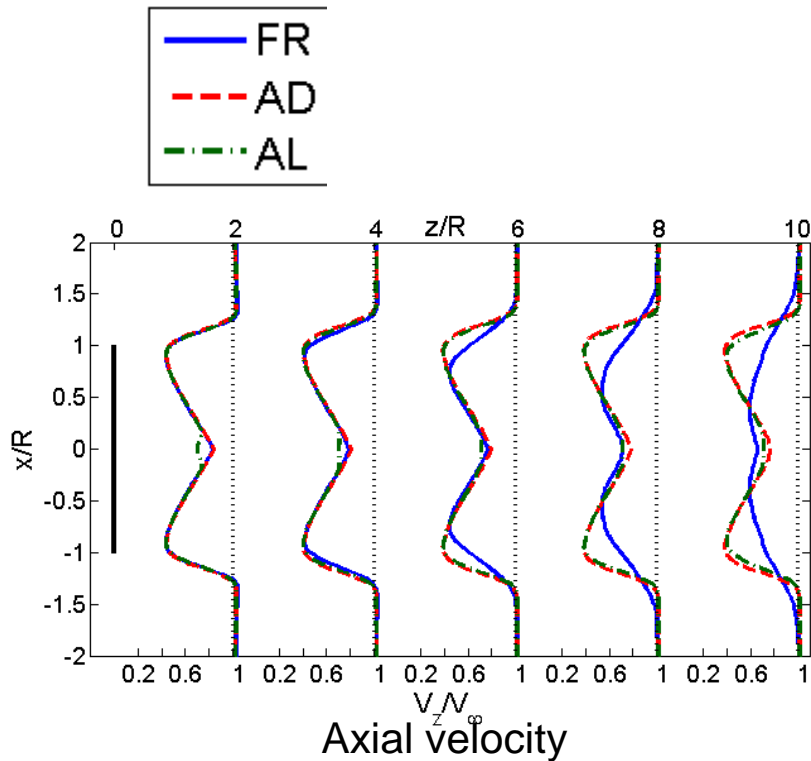
Snapshot of vorticity magnitude contours in horizontal cross-section through rotor center.

Results

Test cases

- Uniform inflow (RANS)
- Uniform inflow (DES)
- Turbulent inflow (DES)

- Good agreement in predicted near wake deficit
- AL and AD in close agreement
- Wake of FR develops faster into a bell shaped form than the AL and AD.



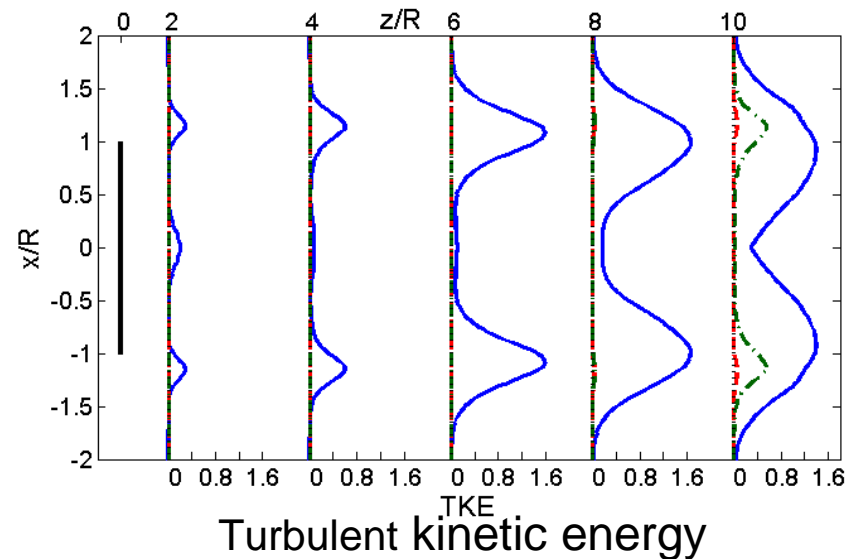
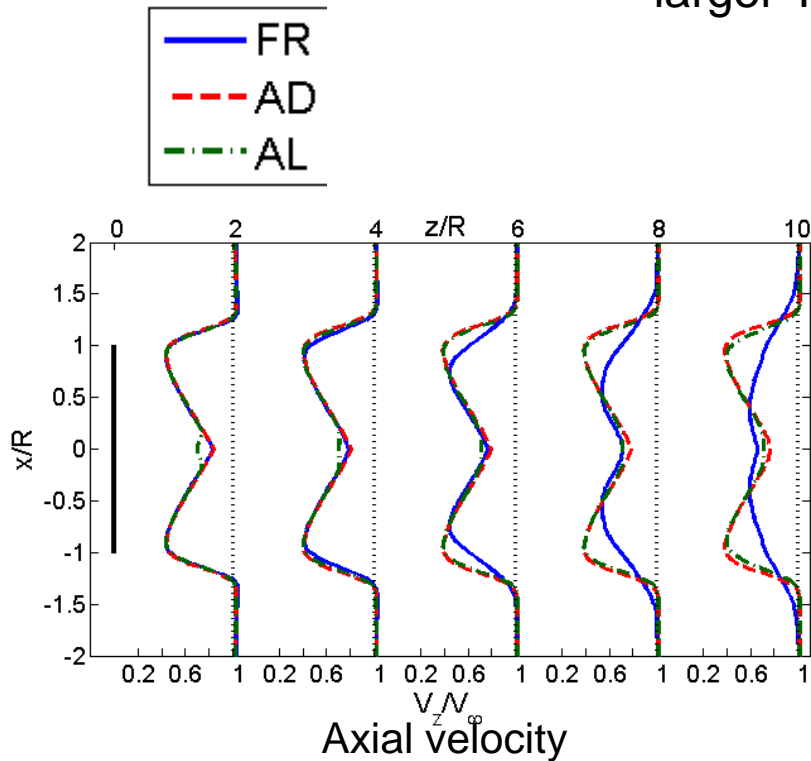
Turbulent kinetic energy

Results

Test cases

- Uniform inflow (RANS)
- Uniform inflow (DES)
- Turbulent inflow (DES)

- Good agreement in predicted near wake deficit
- AL and AD in close agreement
- Wake of FR develops faster into a bell shaped form than the AL and AD.
- Faster spreading of wake is caused by much larger TKE in the FR simulation.



Test cases

- Uniform inflow (RANS)
- Uniform inflow (DES)
- Turbulent inflow (DES)

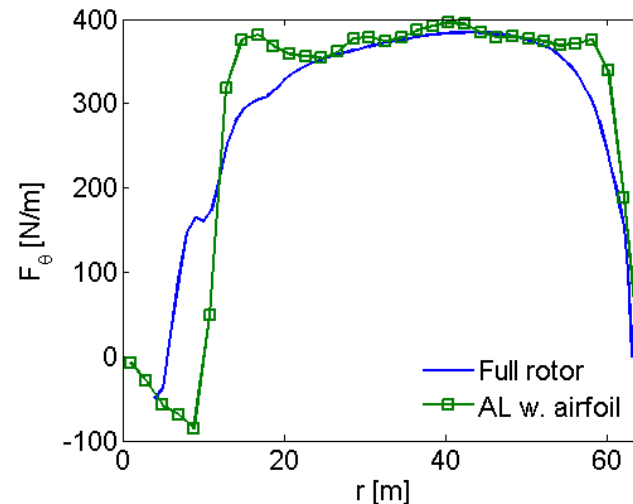
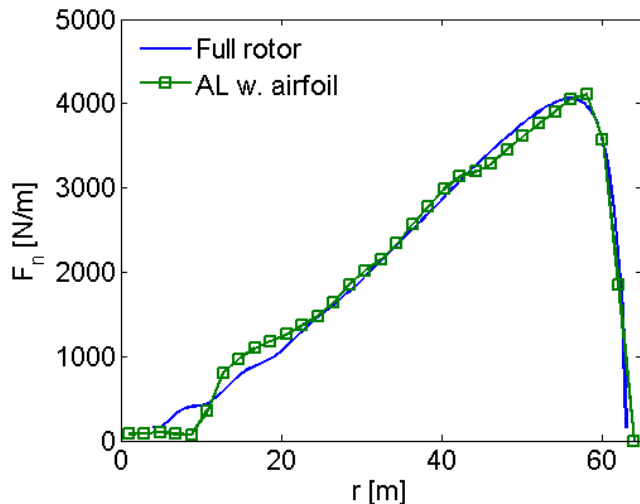
- $V_{\infty} = 8 \text{ m/s}$
- No shear
- Time step $\Delta t = 0.00435\text{s}$ and 0.0244s in FR and AL, respectively.
- Blade loads in AL simulations determined from airfoil data.

Results

Test cases

- Uniform inflow (RANS)
- Uniform inflow (DES)
- Turbulent inflow (DES)

- $V_\infty = 8$ m/s
- No shear
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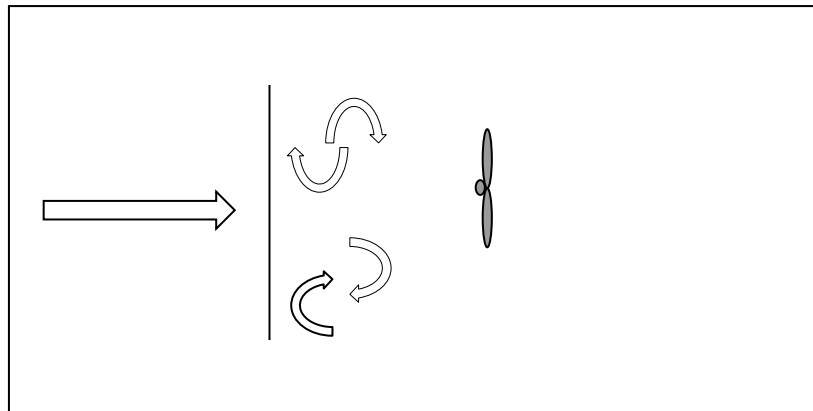


Spanwise distribution of normal and tangential loads on the blades

Test cases

- Uniform inflow (RANS)
- Uniform inflow (DES)
- Turbulent inflow (DES)

- $V_\infty = 8$ m/s
- No shear
- Time step $\Delta t = 0.00435$ s and 0.0244s in FR and AL, respectively.
- Blade loads in AL simulations determined from airfoil data.
- Synthetic atmospheric turbulence inserted in a cross-section 3R upstream of the rotor.
- Turbulence generated using algorithm of Mann.
- Turbulence intensities $TI = 0.015, 0.03$ and 0.06



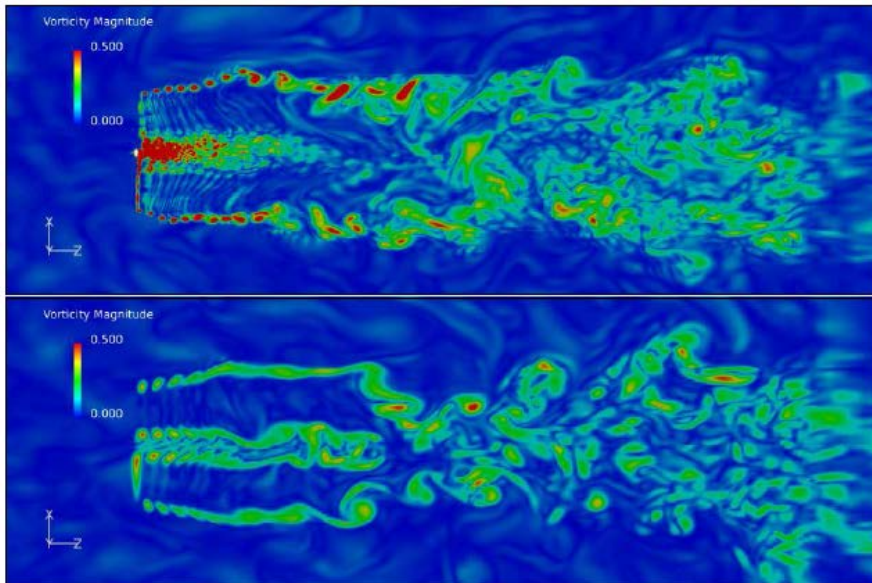
Sketch of computational domain with turbulence cross section upstream of the turbine

Results

Test cases

- Uniform inflow (RANS)
- Uniform inflow (DES)
- Turbulent inflow (DES)

- Ambient turbulence causes wake to break down close to the rotor
- Similar wake behavior predicted by FR and AL



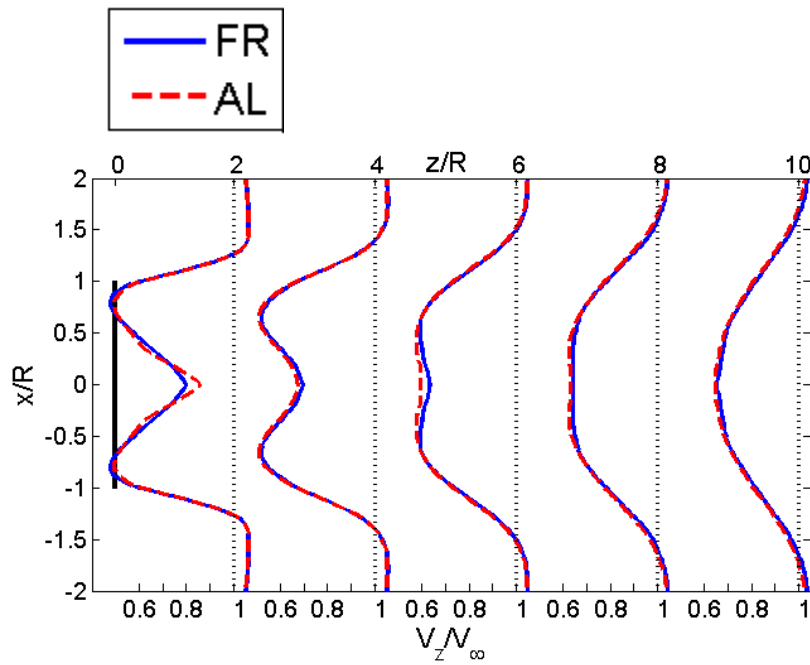
Snapshot of vorticity magnitude contours in horizontal cross-section through rotor center.

Results

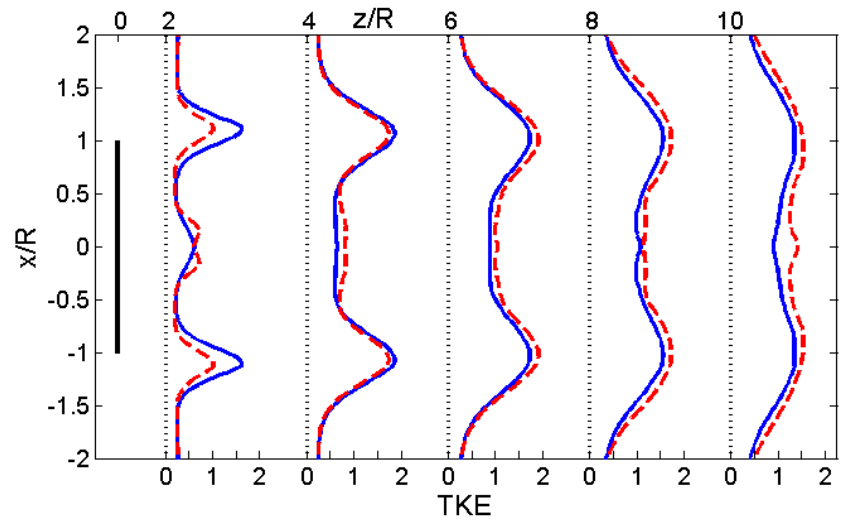
Test cases

- Uniform inflow (RANS)
- Uniform inflow (DES)
- Turbulent inflow (DES)

➤ Wake deficit and TKE predicted by AL in close resemblance with FR when $TI = 0.06$.



Axial velocity



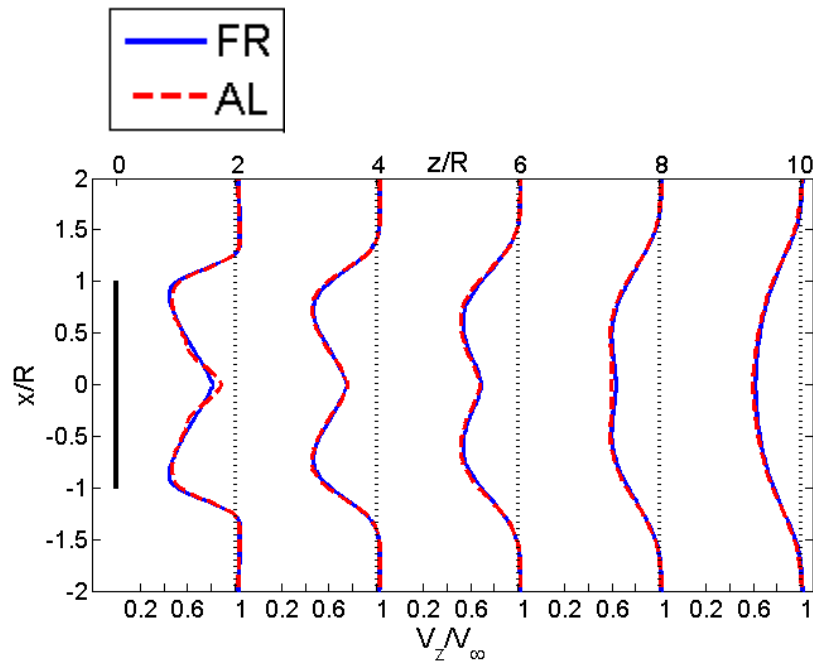
Turbulent kinetic energy

Results

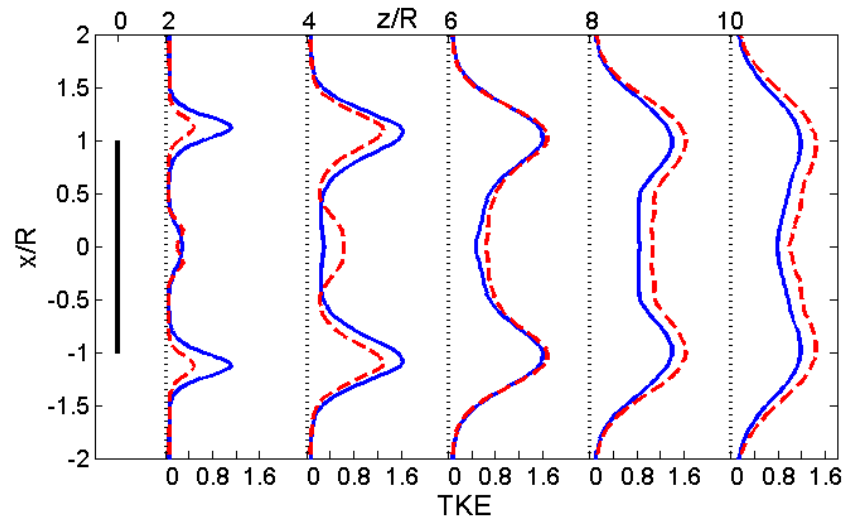
Test cases

- Uniform inflow (RANS)
- Uniform inflow (DES)
- Turbulent inflow (DES)

- Wake deficit and TKE predicted by AL in close resemblance with FR when $TI = 0.06$.
- and when $TI = 0.03$



Axial velocity



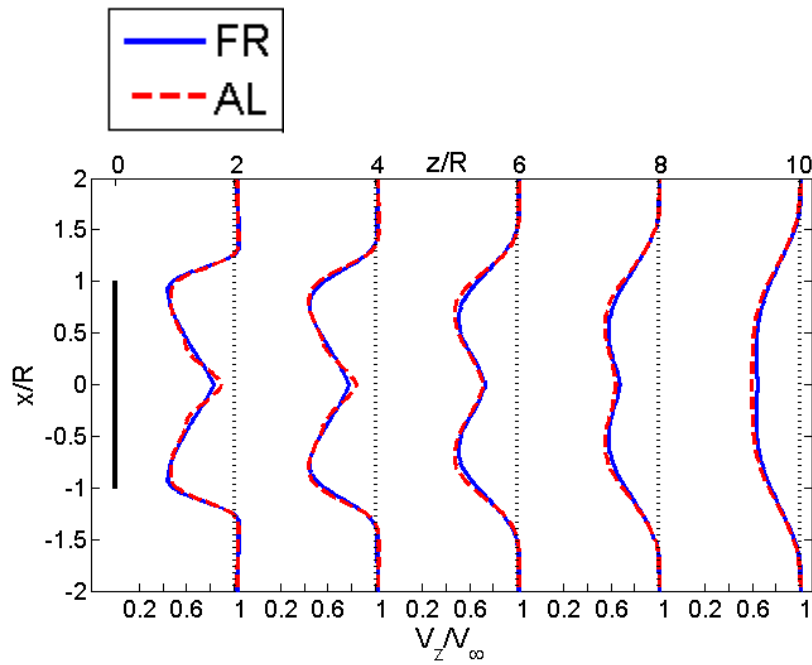
Turbulent kinetic energy

Results

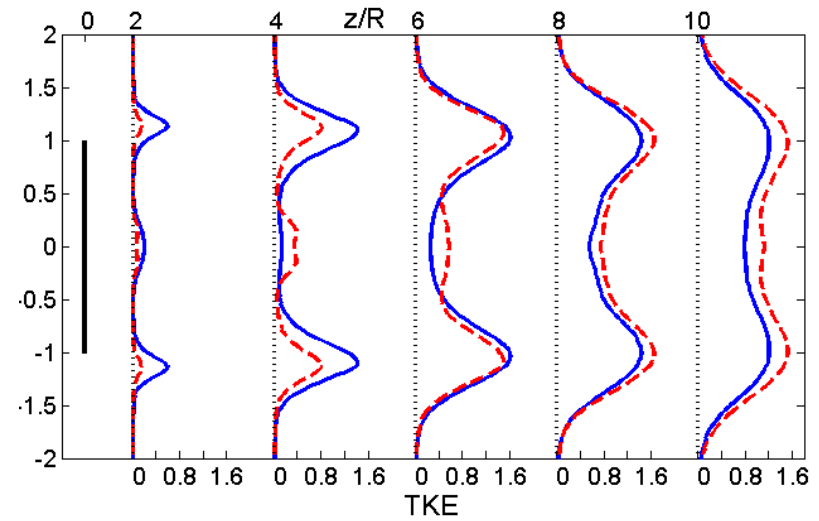
Test cases

- Uniform inflow (RANS)
- Uniform inflow (DES)
- Turbulent inflow (DES)

- Wake deficit and TKE predicted by AL in close resemblance with FR when $TI = 0.06$.
- and when $TI = 0.03$
- and when $TI = 0.015$



Axial velocity



Turbulent kinetic energy

- Uniform and laminar inflow
 - Three models show good agreement in axial velocity up to 2D downstream of the turbine.
 - Further downstream the FR simulation predicts a faster smearing of the mean gradients
 - Much higher TKE in the FR simulation
 - Generally good agreement between AL and AD for all downstream position.

- Turbulent inflow
 - Good resemblance between wake behavior predicted using FR and AL at all TI.

Future works



- AD in turbulent inflow

- Non-uniform inflow
 - Shear
 - Yaw
 - Wake

- Grid dependency study

Turbulent inflow

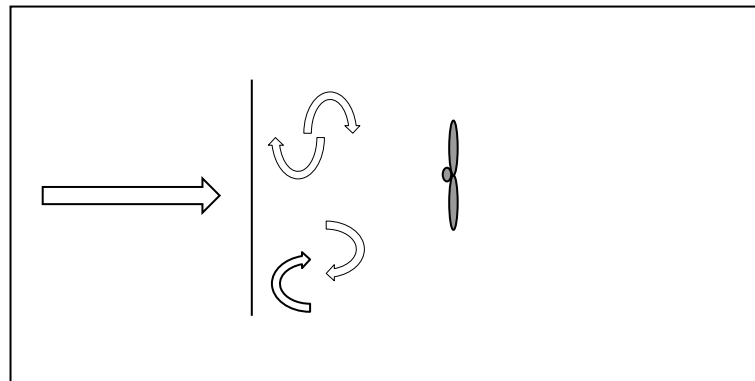
Atmospheric turbulence model:

- Synthetic turbulent fluctuations introduced in a cross section upstream of the turbine.
- Turbulence field generated using the method of Mann.
- Fluctuations introduced through unsteady body forces, \mathbf{f} , in the cross section (corresponding to unsteady actuator disc).

$$\mathbf{f} = \dot{m}\mathbf{u}$$

- Forces smeared in the direction normal to the cross section to avoid singular behaviour.

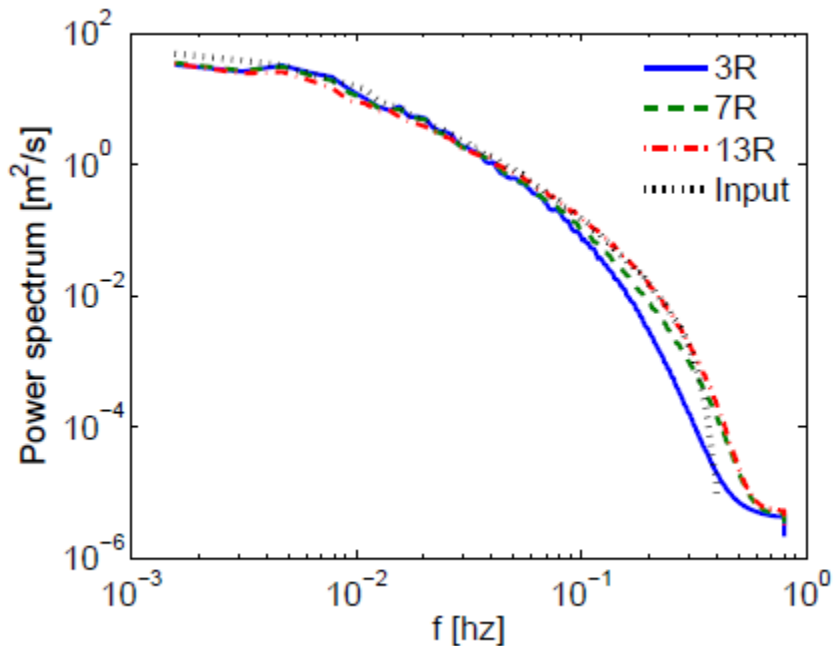
$$\mathbf{f}_\varepsilon = \mathbf{f} \otimes \eta_\varepsilon, \quad \eta_\varepsilon = \frac{1}{\varepsilon\pi^{1/2}} \exp\left[-\frac{d^2}{\varepsilon^2}\right]$$



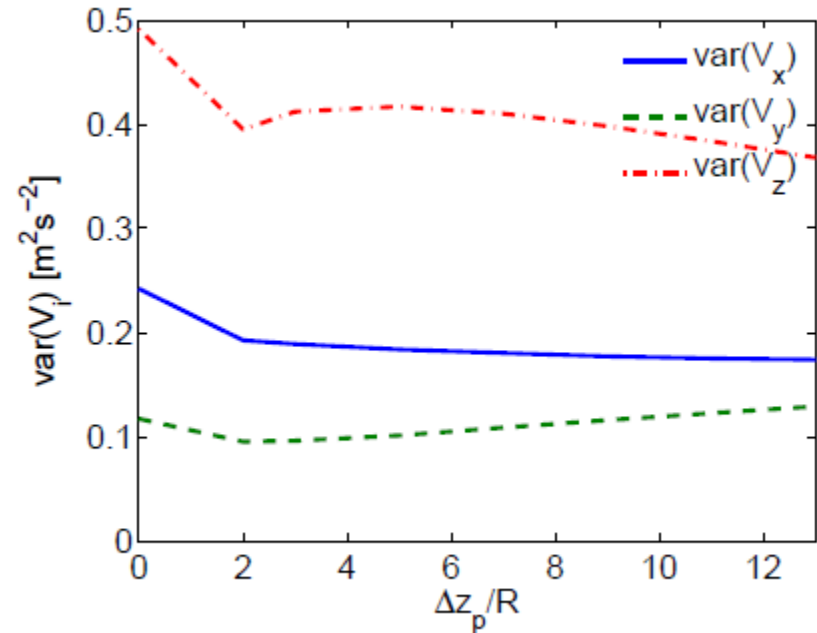
Turbulent inflow

Decay of inserted turbulence:

- Turbulence decays (no production to balance dissipation)
- Decay of energy is in the order of 80% in the region of interest



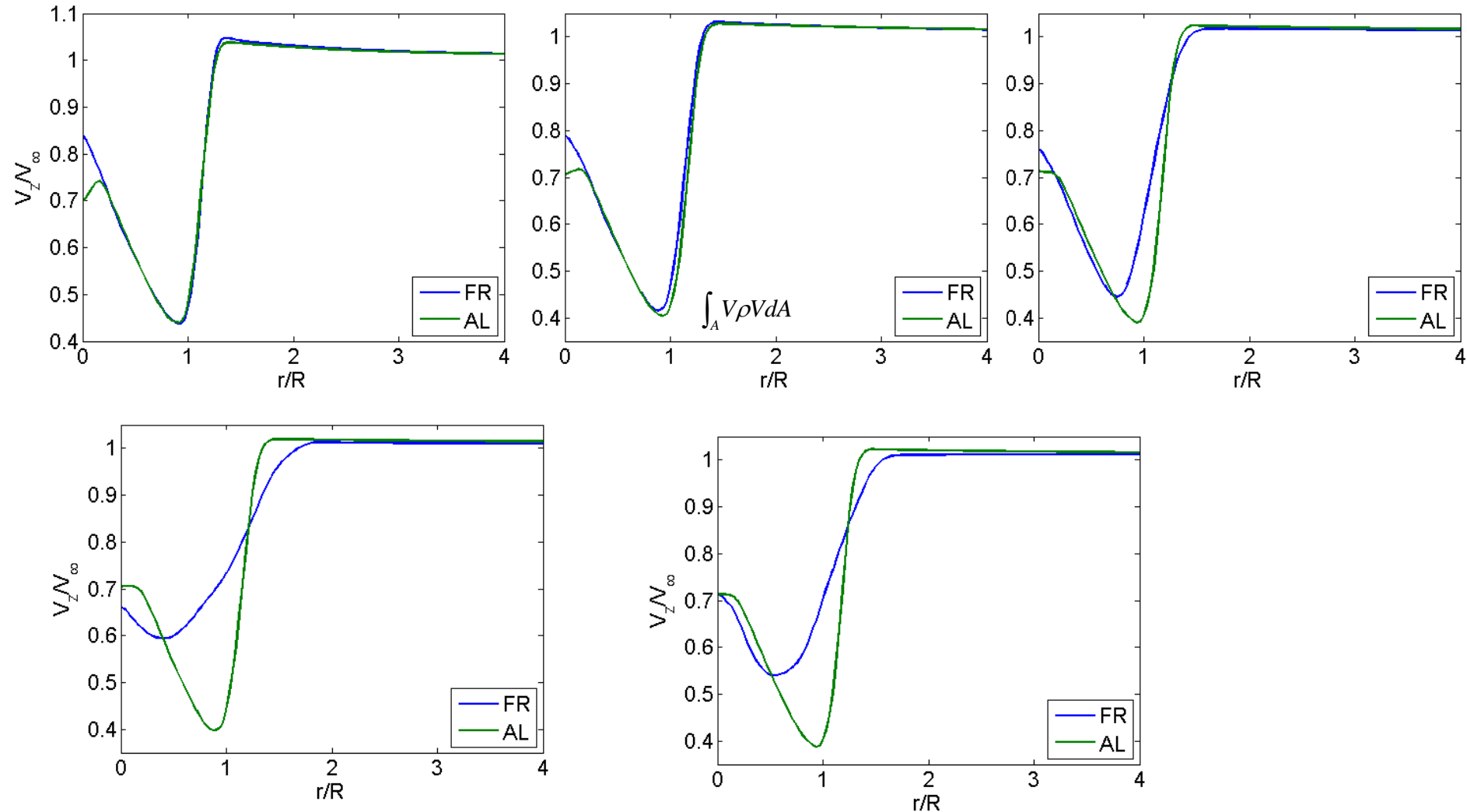
Spectra of axial velocity downstream of cross-section



Variance of each velocity component downstream of cross-section

Velocity profiles

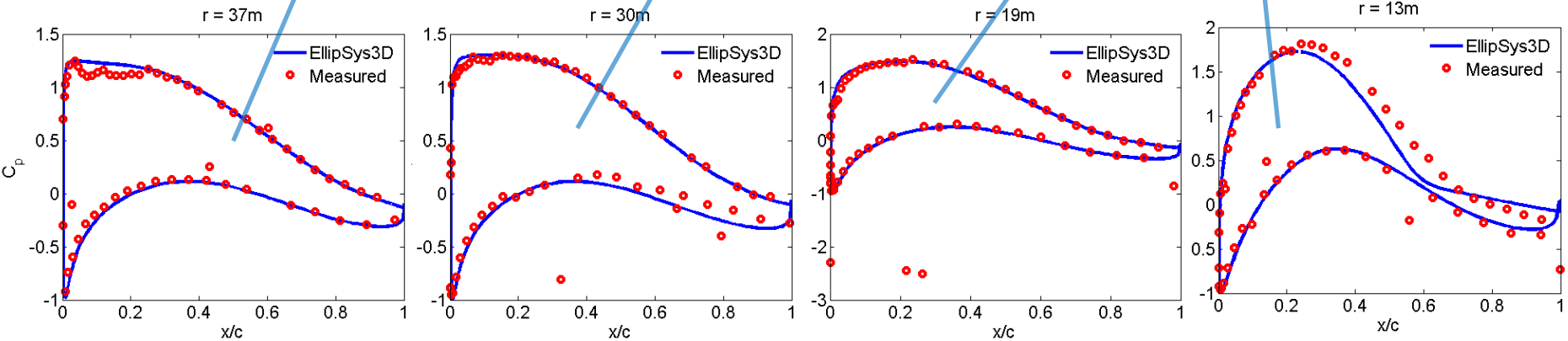
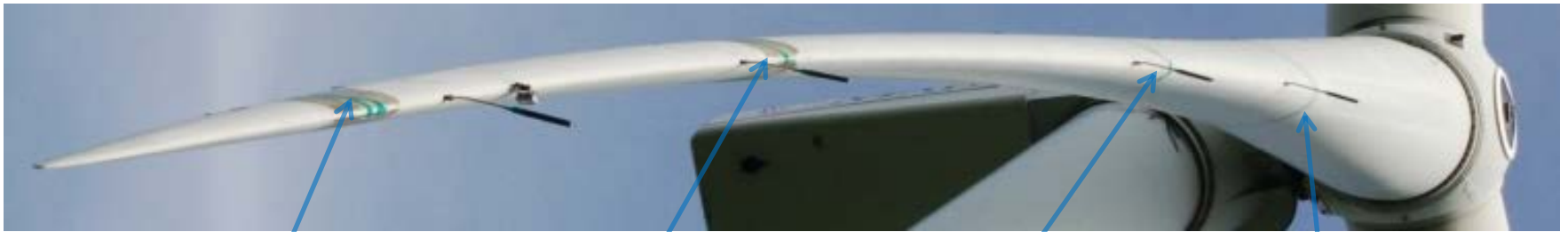
➤ Integration of momentum over circular cross-section with $r=2R$ gives



Numerical investigations

Wind turbine models in CFD

- Fully resolved rotor (FR)
- Actuator line model (AL)
- Actuator disc model (AD)



Comparison of computed and measured pressure distributions obtained during the DAN-AERO project

Test cases

- Uniform inflow (RANS)
- Uniform inflow (DES)
- Turbulent inflow (DES)

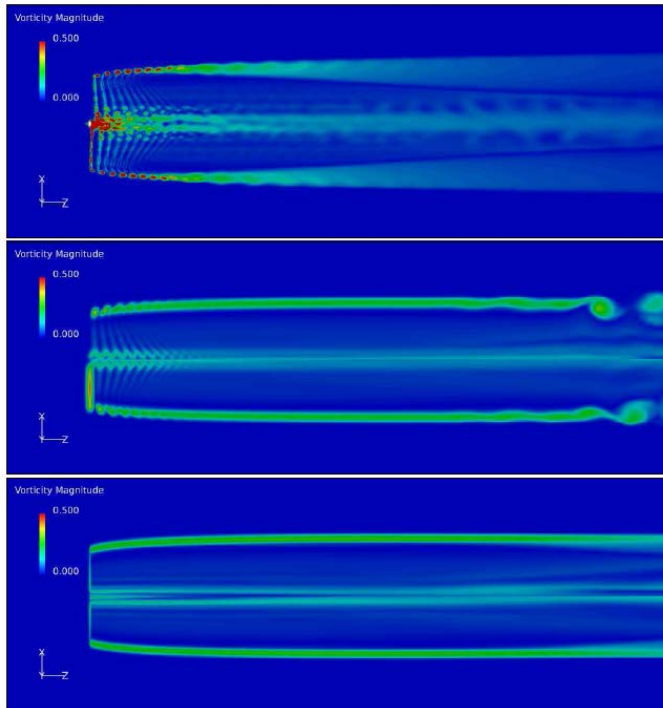
- $V_\infty = 8 \text{ m/s}$
- Closure using k- ω SST turbulence model
- Steady state AD simulation
- $\Delta t = 0.00435\text{s}$ and 0.0244s for FR and AL, respectively (1500 and 267 time steps per rev.)
- Forces from FR simulation applied directly in the AL and AD simulations
- Computing expenses:
 - $T_{\text{FR}} \approx 21T_{\text{AL}} \approx 105T_{\text{AD}}$

Results

Test cases

- Uniform inflow (RANS)
- Uniform inflow (DES)
- Turbulent inflow (DES)

- Vorticity from tip vortices much stronger in FR than in AL and AD.
- Larger spreading of vorticity in FR
- Similar vorticity contours for AL and AD (except for instability in the far wake)



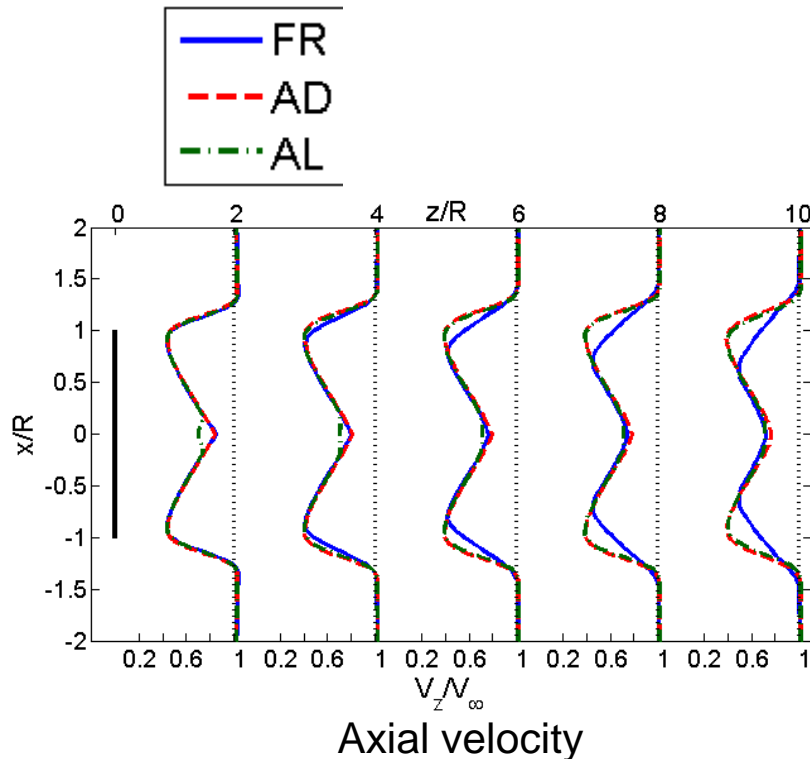
Snapshot of vorticity magnitude contours in horizontal cross-section through rotor center.

Results

Test cases

- Uniform inflow (RANS)
- Uniform inflow (DES)
- Turbulent inflow (DES)

- Good agreement in predicted near wake deficit
- AL and AD in very close agreement
- Faster spreading of wake deficit with FR than with AL and AD.

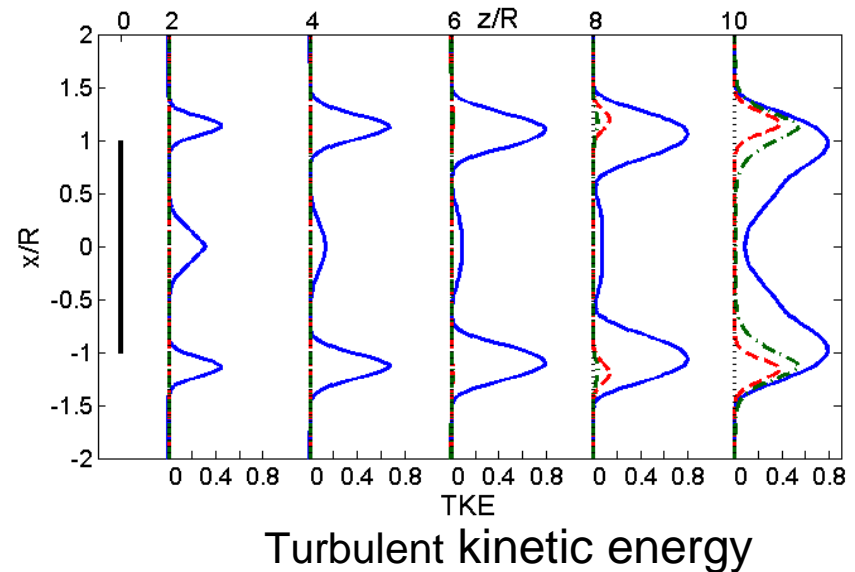
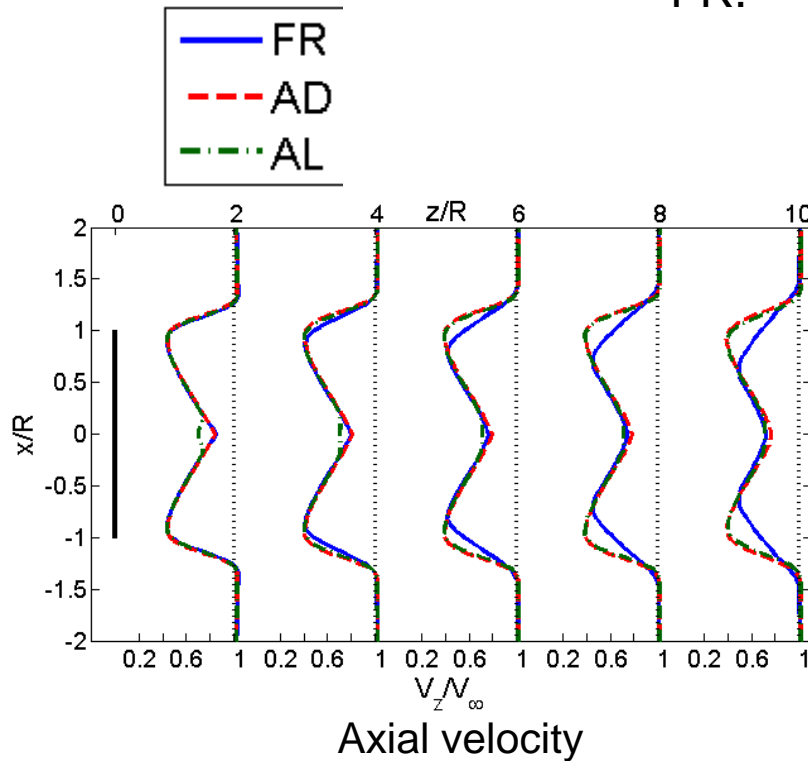


Results

Test cases

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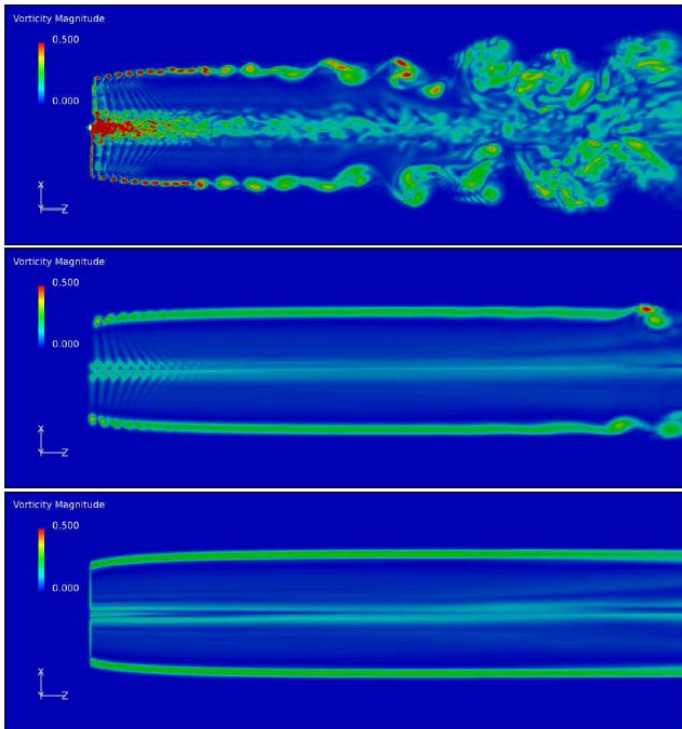
- Good agreement in predicted near wake deficit
- AL and AD in very close agreement
- Faster spreading of wake deficit with FR than with AL and AD.
- Faster spreading caused by much larger TKE in FR.



Test cases

- Uniform inflow (RANS)
- Uniform inflow (DES)
- Turbulent inflow (DES)

- Wake of FR becomes unstable and breaks up.
- Switching to DES has limited effect on AL and AD simulations.
- Reasons for more unstable wake of FR:
 - Higher grid resolution
 - Fluctuating loads (e.g. stall effects near root)



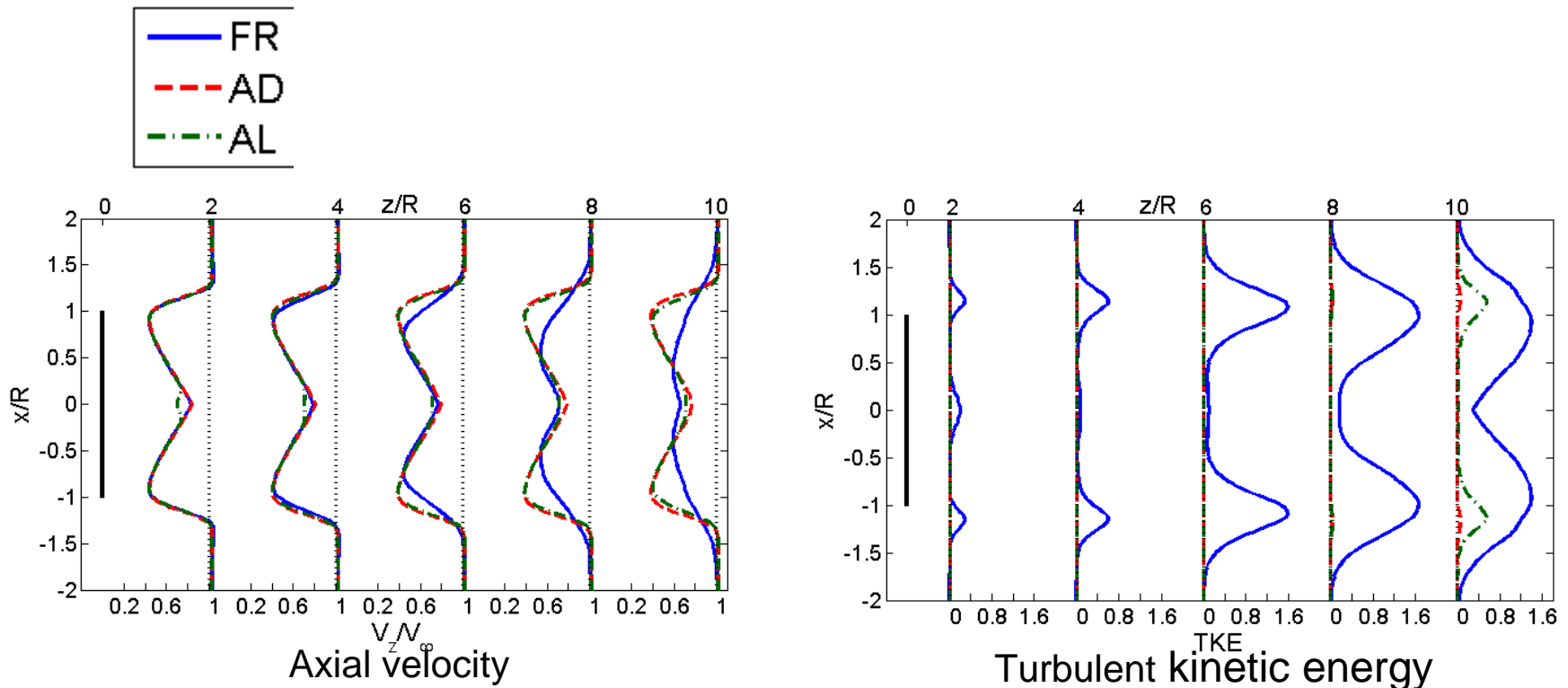
Snapshot of vorticity magnitude contours in horizontal cross-section through rotor center.

Results

Test cases

- Uniform inflow (RANS)
- Uniform inflow (DES)
- Turbulent inflow (DES)

- Wake of FR develops faster into a bell shaped form than the AL and AD.
- AL and AD again in close agreement
- Switching to DES causes increased levels of TKE in the FR simulation



Objectives

- Conduct a consistent comparison of the three models
 - Simulate wake of the NREL 5MW Reference turbine
 - Same numerical setup for all models
- Study the strength and limitations of the models
- Establish guidelines for when and how to use the models

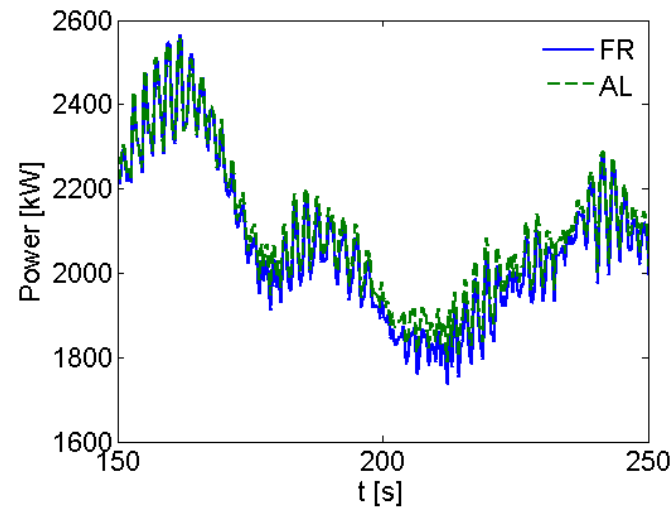
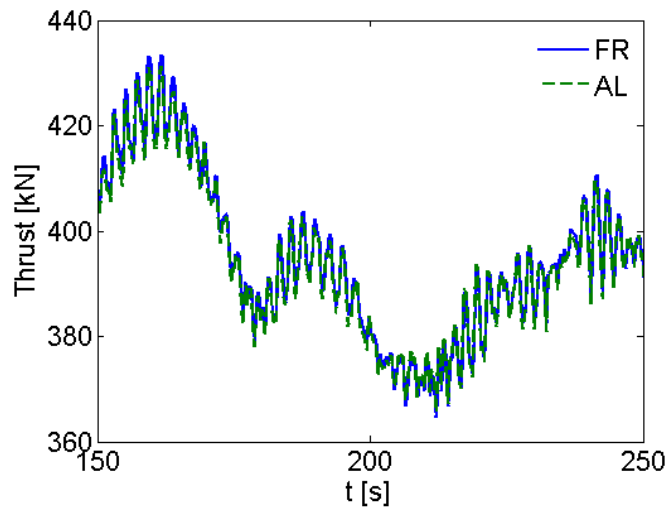


Results

Test cases

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- Uniform inflow (DES)
- Turbulent inflow (DES)

- $V_\infty = 8 \text{ m/s}$
- Time step $\Delta t = 0.00435\text{s}$ and 0.0244s in FR and AL, respectively.
- Blade loads in AL simulations determined from airfoil data.



Temporal variation of integrated thrust and power with FR and AL, respectively