by

#### Jens N. Sørensen

**Department of Mechanical Engineering** 

Technical University of Denmark

## Wake and Wind Farm Aerodynamics

Basic questions and issues:

- How important is the dynamics of the vortex system
- Relationship between strength of the vortices and the blade load
- Conditions for instability
- How far downstream do 'near-wake' and 'far-wake' refer to
- What is the relationship between vortex dynamics and meandering
- How does the added turbulence intensity relate to the loading
- Performance predictions of wind farms
- Life time estimation of turbines in wind farms
- Influence of stability of the atmospheric boundary layer
- Estimation of wind resources in wind farm to wind farm interaction



### Wake and Wind Farm Aerodynamics

#### Simulation models

- Momentum theory (Frandsen)
- Linearized Navier-Stokes (Fuga)
- Parabolised Navier-Stokes (Ainslie, UMPWAKE)
- Reynolds Averaged Navier-Stokes (RANS)
- Detached Eddy Simulation (DES)
- Large Eddy Simulation (LES)
- Actuator Disc/Line LES (AD/L-LES)

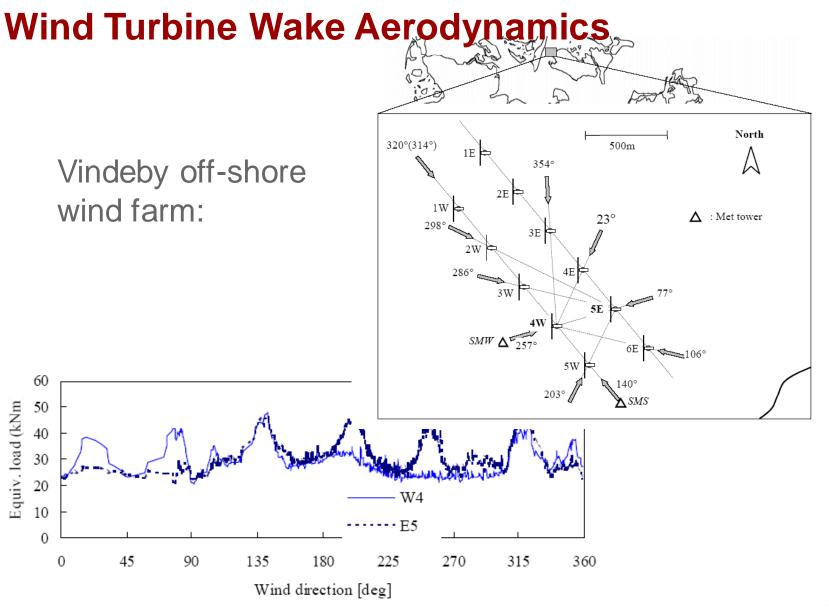


#### Wind Turbine Wake Aerodynamics

Horns Rev offshore wind farm:







**Department of Mechanical Engineering** Technical University of Denmark

# 

#### The actuator line technique

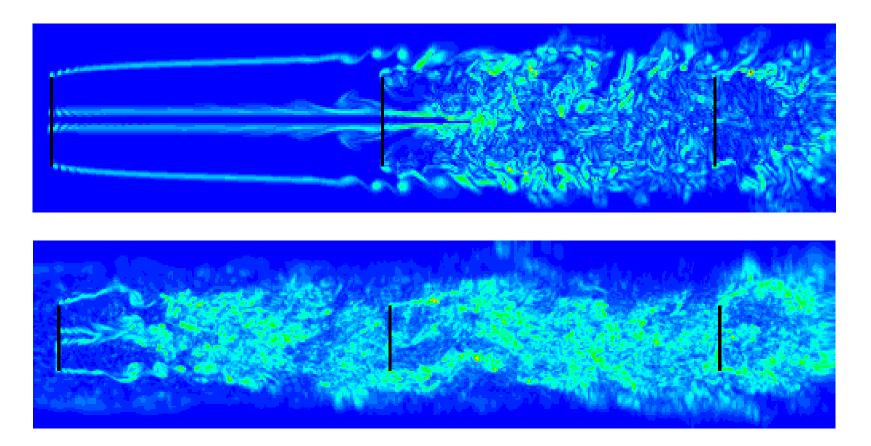
Basic idea: • Replace rotor blades by body forces

- Determine body forces from aerofoil data
- Simulate flow domain using DNS or LES
- Computing code: *EllipSys3D*



Danmarks Tekniske Universitet

#### Vortex structures in the wake of a row of rotors



Development of wake behind three rotors in a row at  $W_0 = 10$  m/s; Turbine spacing 6 rotor radii. A) Constant inflow; B) Turbulent inflow.

DTU

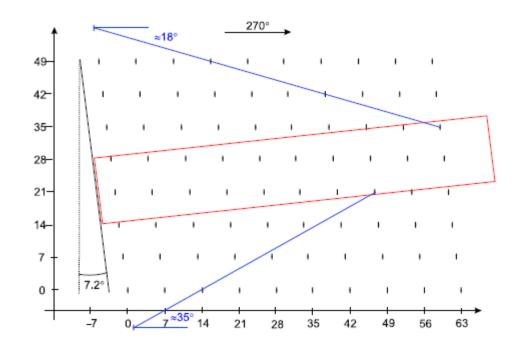


FIGURE 1. Layout of Horns Rev Wind Farm

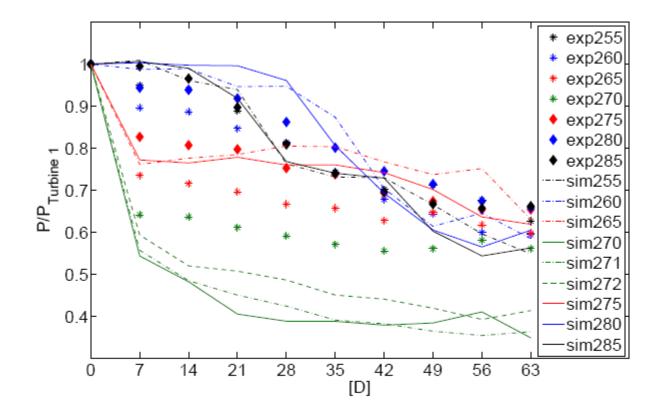
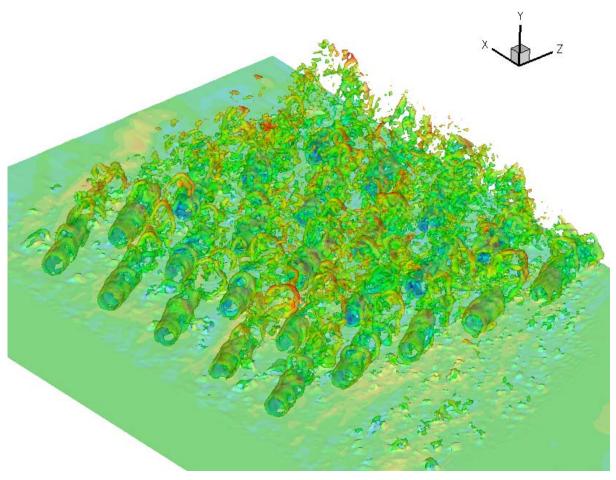


FIGURE 13. Simulation results compared with measurements. Results from both simulations and measurements are shown for inflow angles between 255 and 285 degrees, i.e.,  $\pm$  15 degrees from the westerly direction.

#### Modelling of Turbulent and Atmospheric Turbulence



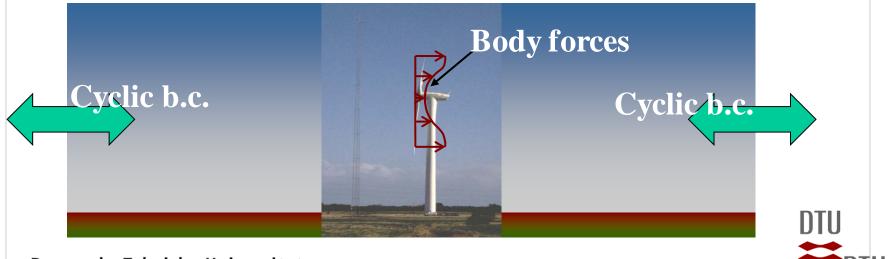
Vorticity shed from 5x5 turbines in a farm computed by actuator disk method

DTU

#### Simulation of turbulence inside wind farm

#### Basic idea: • Replace rotor blades by body forces

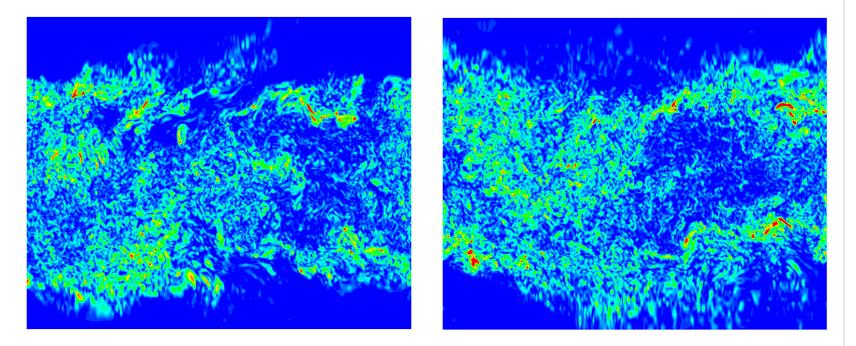
- Determine body forces from aerofoil data
- Simulate an 'infinite' row of turbines using cyclic boundary conditions



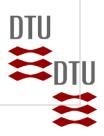
#### **Danmarks Tekniske Universitet**

#### Simulation of turbulence inside wind farm

Cross sectional turbulent flow fields:

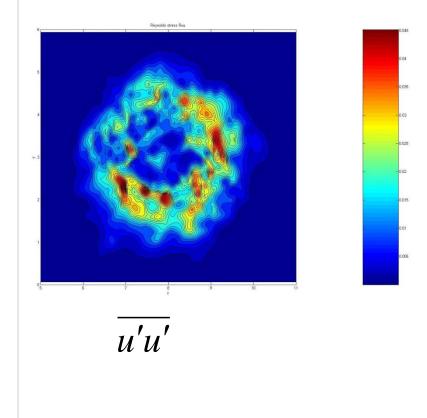


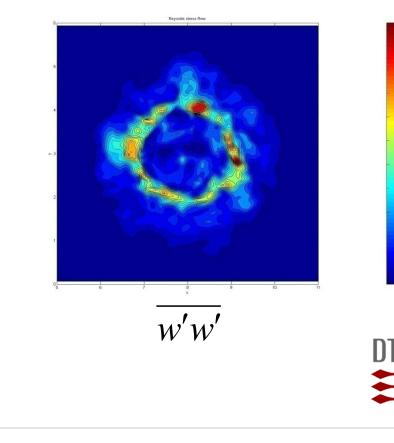
Iso-vorticity contours in the final stage



#### Simulation of turbulence inside wind farm

#### Reynolds stresses:





ABL flow equations:

DTU

Dynamic Sub-Grid Scale model:

**SGS stress:**  $\tau_{ij} - \frac{1}{3} \tau_{kk} \delta_{ij} = -2 v_{sgs} \widetilde{S}_{ij}$ **SGS heat flux:**  $q_j = -\frac{v_{sgs}}{Pr_{sgs}} \frac{\partial \widetilde{\theta}}{\partial x_j}$ 

Pr<sub>sgs</sub> is the SGS Prandtl number

**Resolved strain rate:**  $\widetilde{S}_{ij} = (\partial \widetilde{u}_i / \partial x_j + \partial \widetilde{u}_j / \partial x_i)/2$ 

**Smagorinsky mixing length model:**  $v_{\text{sgs}} = C_s^2 \widetilde{\Delta}^2 |\widetilde{S}|$ , where  $|\widetilde{S}| = (2\widetilde{S}_{ij}\widetilde{S}_{ij})^{1/2}$ 

**Problem:** C<sub>s</sub> and Pr<sub>sgs</sub>?

Dynamic SGS model: (Germano (1991); Lilly (1992))

# Least squares minimization of error:

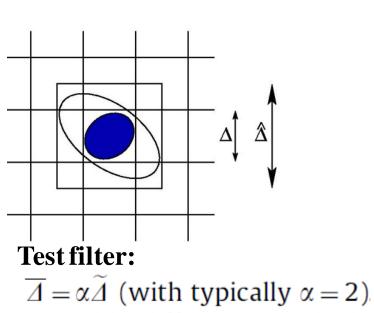
$$C_s^2(\widetilde{\Delta}) = \frac{\langle L_{ij}M_{ij}\rangle_{\mathcal{L}}}{\langle M_{ij}M_{ij}\rangle_{\mathcal{L}}},$$

$$C_s^2 Pr_{sgs}^{-1}(\widetilde{\Delta}) = \frac{\langle K_i X_i \rangle_{\mathcal{L}}}{\langle X_i X_i \rangle_{\mathcal{L}}}$$

#### Scale dependence parameters:

 $\beta = C_s^2(\alpha \widetilde{\Delta}) / C_s^2(\widetilde{\Delta})$  $\beta_{\theta} = C_s^2 P r_{sgs}^{-1}(\alpha \widetilde{\Delta}) / C_s^2 P r_{sgs}^{-1}(\widetilde{\Delta})$ 

**Typically:**  $\beta = 1$  and  $\beta_{\theta} = 1$ 



Where:

$$L_{ij} = \overline{\widetilde{u}_{i}\widetilde{u}_{j}} - \overline{\widetilde{u}_{i}}\overline{\widetilde{u}_{j}}$$

$$M_{ij} = 2\widetilde{\Delta}^{2}(\overline{|\widetilde{S}|\widetilde{S}_{ij}} - \alpha^{2}\beta|\overline{\widetilde{S}}|\overline{\widetilde{S}}_{ij})$$

$$K_{i} = \overline{\widetilde{u}}\overline{\widetilde{\theta}} - \overline{\widetilde{u}_{i}}\overline{\widetilde{\theta}}$$

$$X_{i} = \widetilde{\Delta}^{2}(\overline{|\widetilde{S}|\partial\overline{\theta}/\partial x_{i}} - \alpha^{2}\beta_{\theta}|\overline{\widetilde{S}}|\partial\overline{\overline{\theta}}/\partial x_{i})$$

$$\Pi \Pi \Pi$$

Dynamic scale-dependent SGS model: (Meneveau et al. (1996); Porte-Agel et al. (2011))

Least squares minimization of error:

$$C_s^2(\widetilde{\Delta}) = \frac{\langle L'_{ij}M'_{ij}\rangle_{\mathcal{L}}}{\langle M'_{ij}M'_{ij}\rangle_{\mathcal{L}}},$$

$$C_s^2 Pr_{sgs}^{-1}(\widetilde{\Delta}) = \frac{\langle K'_i X'_i \rangle_{\mathcal{L}}}{\langle X'_i X'_i \rangle_{\mathcal{L}}}$$

Scale dependence parameters:

$$\beta = \frac{C_s^2(\alpha \widetilde{\Delta})}{C_s^2(\widetilde{\Delta})} = \frac{C_s^2(\alpha^2 \widetilde{\Delta})}{C_s^2(\alpha \widetilde{\Delta})},$$
$$\beta_{\theta} = \frac{C_s^2 P r_{sgs}^{-1}(\alpha \widetilde{\Delta})}{C_s^2 P r_{sgs}^{-1}(\widetilde{\Delta})} = \frac{C_s^2 P r_{sgs}^{-1}(\alpha^2 \widetilde{\Delta})}{C_s^2 P r_{sgs}^{-1}(\alpha \widetilde{\Delta})}$$

Where:

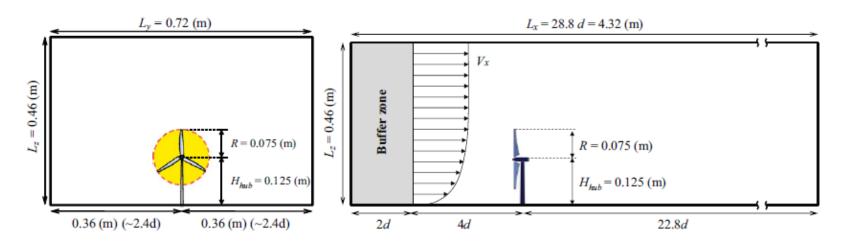
$$\begin{split} L'_{ij} &= \widehat{\widetilde{u}_{i}}\widehat{\widetilde{u}_{j}} - \widehat{\widetilde{u}_{i}}\widehat{\widetilde{u}_{j}} \\ M'_{ij} &= 2\widetilde{\Delta}^{2}(|\widehat{\widetilde{S}}|\widehat{\widetilde{S}}_{ij} - \alpha^{2}\beta^{2}|\widehat{\widetilde{S}}|\widehat{\widetilde{S}}_{ij}) \\ K'_{i} &= \widehat{\widetilde{u}}\widehat{\widetilde{\theta}} - \widehat{\widetilde{u}_{i}}\overline{\widetilde{\widetilde{\theta}}} \\ X'_{i} &= \widetilde{\Delta}^{2}(|\widehat{\widetilde{S}}|\widehat{\partial\widetilde{\theta}}/\partial x_{i} - \alpha^{2}\beta^{2}_{\theta}|\widehat{\widetilde{S}}|\widehat{\partial\widetilde{\theta}}/\partial x_{i}) \end{split}$$

Second test filter:  $\widehat{\varDelta} = \alpha^2 \widetilde{\varDelta}$ 

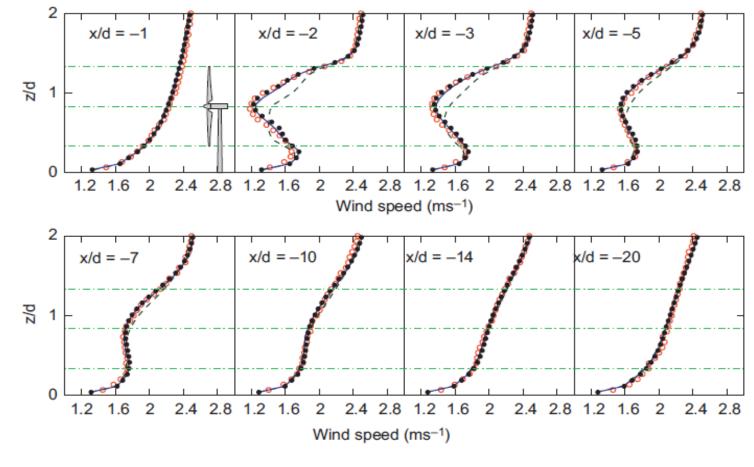


Some results of ABL-LES computations: (Porte-Agel et al. (2011))

#### **Flow domain**

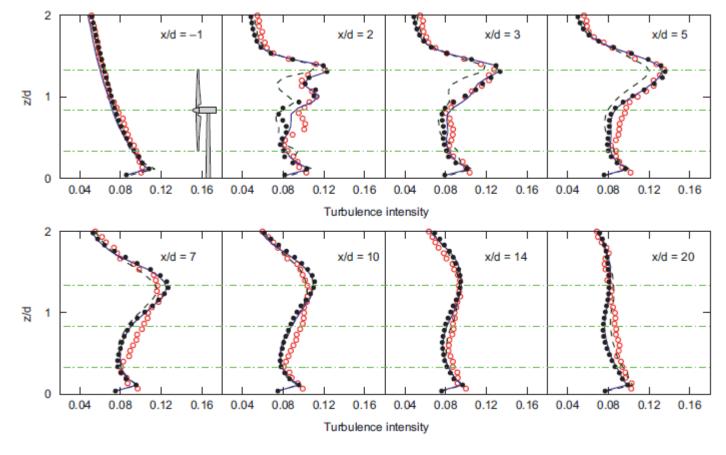


Comparison of LES computations with measurements (Porte-Agel et al. (2011))



DTU

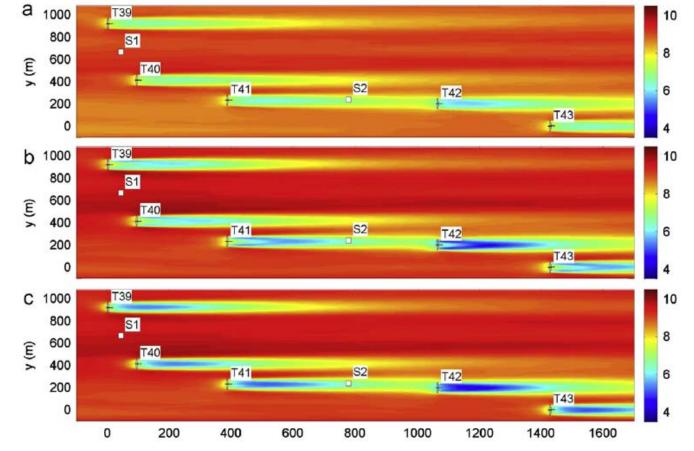
Comparison of LES computations with measurements (Porte-Agel et al. (2011))



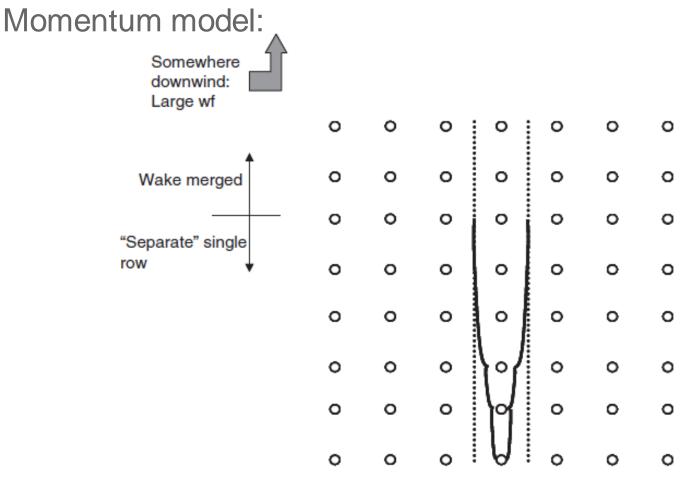
DTU

**Department of Mechanical Engineering** Technical University of Denmark

Comparison between actuator disc and actuator line models (Porte-Agel et al. (2011)):



DTU



From Frandsen et al.: Wind Energy vol. 9, 2006



Momentum model:

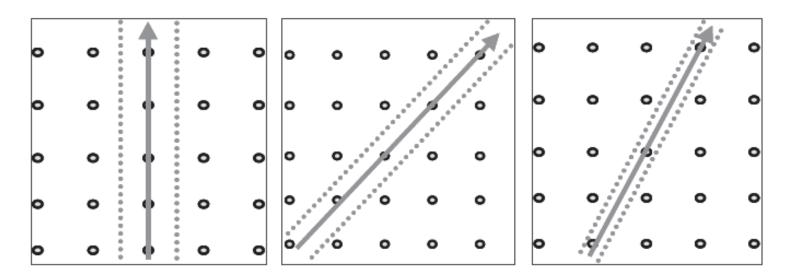
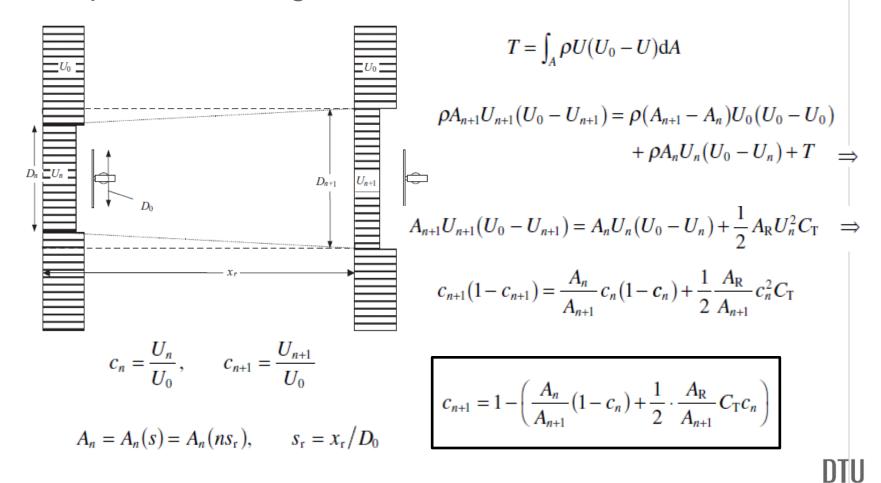


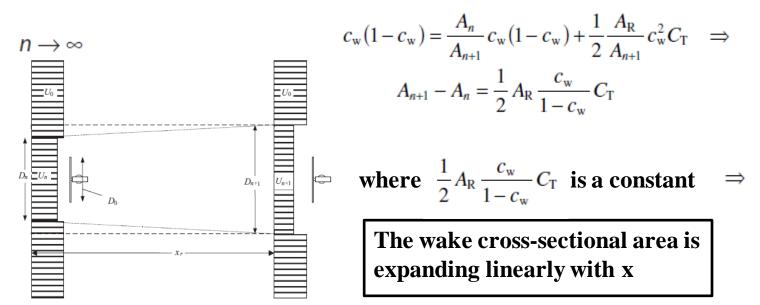
Figure 2. Examples of wind turbine patterns for different wind directions

DTU

Multiple wakes, single row:



Momentum model:



Frandsen model:  $A_{n+1} - A_n = \frac{\pi}{4} D_0^2 [\beta + \alpha s_r (n+1)] - \frac{\pi}{4} D_0^2 (\beta + \alpha s_r n) = A_R \alpha s_r$ where  $\alpha = \frac{1}{2} \frac{C_T}{s_r} \frac{c_w}{1 - c_w}$ 



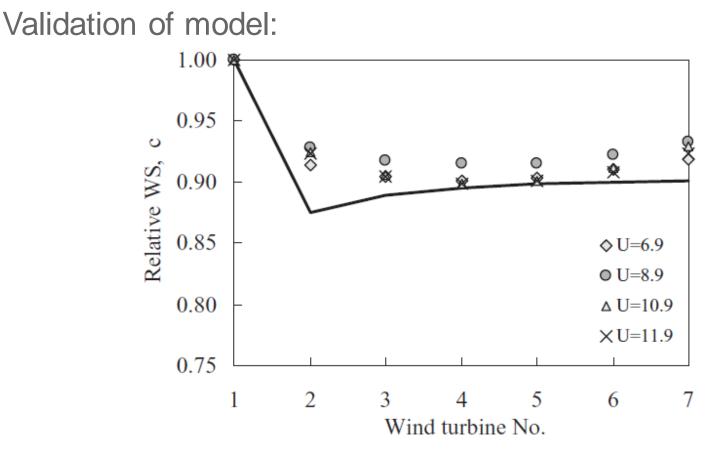
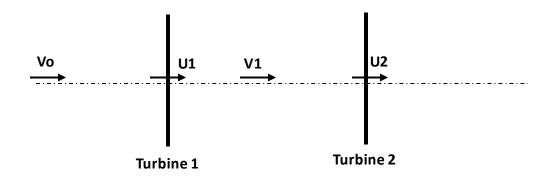


Figure 6. Measurement of wind speed ratio  $c_i$  at Nørrekær Enge II. Wind speeds are derived from power signals. Average is taken over six rows, each with seven units.  $s_r \approx 7$ . The wind farm consists of 42,300 kW units

Assignment:

Determine the maximum performance of two turbines in tandem:



Assume that  $U1 = \frac{1}{2}(V0 + V1)$ , and introduce a1 = 1 - U1/V0and a2 = 1 - U2/V1.

- 1. Derive an expression for the power coefficient Cp = Cp(a1,a2)
- 2. What is the optimum Cp and operating conditions for the rotor system
- 3. Can the model be extended to an arbitrary number of wind turbines

